

1 Shape-Morphing Metamaterials

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11 ABSTRACT

Mechanical metamaterials leverage geometric design to achieve unconventional properties, such as high strength at low density, efficient wave guiding, and complex shape morphing. The ability to control shape changes builds on the complex relationship between geometry and nonlinear mechanics, and opens new possibilities for disruptive technologies across diverse fields, including wearable devices, medical technology, robotics, and beyond. In this review of shape-morphing metamaterials, we examine the current state of the field and propose a unified classification system for the mechanisms involved, as well as the design principles underlying them. Specifically, we explore two main categories of unit cells—those that exploit structural anisotropy or internal rotations—and two potential approaches to tessellating these cells: based on kinematic compatibility or geometric frustration. We conclude by discussing the available design tools and highlighting emerging challenges in the development of shape-morphing metamaterials.

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25 1 Introduction

26 All around us, things change shape, often in a highly controlled fashion; think of the blooming of a flower, the
27 intricate folding of a protein, the expansion of a deployable shelter, or the purposeful movement of a walking
28 robot. The ability to program specific shape changes into structures opens up numerous compelling applications in
29 deployable and reconfigurable materials and structures [1–4], robotics [5–7], biomedical devices [8, 9], and for the
30 control of light fields [10–13]. In this review, we focus on a specific subset of shape-morphing structures, namely
31 shape-morphing metamaterials, a class of mechanical metamaterials capable of undergoing complex reconfiguration.
32 Mechanical metamaterials [14, 15]—carefully structured materials with mechanical properties governed by structure,
33 where geometry plays a key role, rather than composition—present a promising platform for realizing such controlled
34 shape changes. While initial efforts in the field focused on achieving unconventional elastic properties such as
35 auxetic response [16], more recently, the research community has been exploring opportunities to embed nonlinear,
36 non-periodic deformations into mechanical metamaterials, leading to targeted shape changes.

37 In this review, we concentrate on shape-morphing metamaterials—a specific subclass of mechanical metamaterials—
38 and refer readers to numerous other reviews on mechanical metamaterials for a broader perspective [14, 15, 17–25].
39 Our goal is to unify recent literature, to offer a pedagogical introduction for researchers interested in shape-changing
40 metamaterials, to highlight the field’s main achievements, and to identify the remaining challenges. We address the
41 following questions: What types of shape changes can metamaterials achieve? What are the different classes of
42 shape-changing metamaterials? What principles enable the design of target shape changes?

43 Shape-morphing metamaterials require careful unit cell design, robust patterning strategies, and reliable actuation
44 mechanisms to achieve desired shape transformations. While recent reviews have thoroughly discussed actuation
45 methods for mechanical metamaterials (e.g. osmotic shocks, magnetic forces, hydrostatic force, etc)[18, 26–28],
46 here we focus on providing a systematic overview of the design principles that enable complex shape-morphing. To
47 this end, we first present the fundamental working principles of unit cells used in shape-changing metamaterials.
48 We then survey the strategies for assembling unit cells into architected materials capable of achieving prescribed
49 shape transformations. Finally, we briefly review design tools for solving the inverse problem of identifying suitable
50 metamaterial geometries for specific target shapes. We conclude with an outlook on the challenges and opportunities
51 that lie ahead in advancing the field of shape-morphing metamaterials.

52 2 Unit cell design principles

53 The unit cells in shape-changing metamaterials proposed to date can be classified into two main categories: those
54 based on anisotropic responses and those based on internal rotations. Anisotropic unit cells are typically made
55 with elongated structural elements, like fibers and tubes, where axial deformations are energetically unfavorable.
56 In contrast, rotation-based unit cells consist of stiff structural elements connected by slender joints, and their
57 deformation is primarily driven by rotations about these joints, leading to significant changes in volume.

58 2.1 Unit cells based on anisotropic mechanism

59 Nature provides many examples of thin objects for which the anisotropy induced by the orientations of internal fibers
60 results in complex shape changes. These include the hygroscopic actuation of wheat awns [29], the formation of the
61 chiral shape of some seed pods [30], and the opening of pine cones [31]. Through bioinspiration and biomimicry,
62 many designs of anisotropic unit cells have been proposed. As illustrated in Fig. 1, four primary categories of
63 anisotropic unit cells have been utilized to create shape-morphing metamaterials: fiber-elastomer composites, knits,
64 tubes, and anisotropic foams. We briefly discuss these categories below. Note that although shape-morphing typically
65 involves large deformations of the unit cell, in the following sections we also introduce closed-form predictions
66 in the linear regime to provide intuition about the underlying mechanisms. While these linear predictions are not
67 expected to quantitatively capture the full nonlinear response of the unit cells, they offer valuable insights into the
68 fundamental principles at play.

69 *Fiber-elastomer composites.* — Fiber-elastomer composites consist of stiff fibers embedded in a soft matrix [32–35].
70 Although these composites typically undergo large deformations that quantitatively require a nonlinear analysis,
71 their key behavioral characteristics can be captured by a simple orthotropic linear elastic model. In two dimensions
72 (under plane stress conditions), this model relates the strain, ϵ_{ij} , and stress, σ_{ij} , components as

$$\begin{bmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \epsilon_{12} \end{bmatrix} = \begin{bmatrix} \frac{1}{E_1} & -\frac{\nu_{12}}{E_1} & 0 \\ -\frac{\nu_{12}}{E_1} & \frac{1}{E_2} & 0 \\ 0 & 0 & \frac{2}{G} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix}, \quad (1)$$

73 where E_1 and E_2 denote the Young's modulus along the x_1 and x_2 directions, ν_{12} represents the Poisson's ratio and
74 G is the shear modulus. For a composite with fibers oriented along the x_1 direction (see Fig. 1a) $E_1 \gg E_2$, since
75 the fibers resist deformations along their axis. It follows that a simple hydrostatic loading, $\sigma_{ij} = P\delta_{ij}$, leads to
76 deformations perpendicular to the fibers larger than those along the fibers ($\epsilon_{11} < \epsilon_{22}$), since $\epsilon_{11} = P(1/E_1 - \nu_{12}/E_1)$
77 and $\epsilon_{22} = P(1/E_2 - \nu_{12}/E_1)$. In planar structures, this behavior results in a pronounced out-of-plane deformation
78 that can be controlled by tuning the orientation of the fibers [35]. Additionally, in cylindrical structures by varying
79 the fiber angle a broad spectrum of deformations can be induced upon inflation, such as axial extension, radial
80 expansion, and twisting [32, 34, 36–40].

81 *Knits.* — Knitted fabrics are composed of fibers that are interwoven and entangled rather than embedded in a soft
82 matrix (see Fig. 1b). They achieve flexibility through geometrically compliant interlooping stitch architectures,
83 even when the yarns themselves are inextensible [41–48]. Additionally, these fabrics can be self-deforming if made
84 using functional yarns [42, 49–51]. Knitted fabrics are inherently anisotropic due to their looped structure and hold
85 promise for creating shape-morphing metamaterials, but few practical examples have been reported to date [52].

86 *Tubes.* — Thin elastic tubes made of isotropic elastic materials deform in an anisotropic manner upon inflation (see
87 Fig. 1c). For a thin and long inflated elastic tube with a radius R and thickness t , the strains in the linear elastic
88 regime along the circumferential and longitudinal directions are given by [53]:

$$\epsilon_t = \frac{PR}{2tE}(1 - 2\nu), \quad \epsilon_c = \frac{PR}{tE}(2 - \nu), \quad (2)$$

89 where P denotes the applied pressure and E and ν are Young's modulus and Poisson's ratio of the material,
90 respectively. Equation (2) illustrates that in inflated elastic tubes, the longitudinal strain is consistently smaller
91 than the circumferential strain, and for an incompressible material ($\nu = 0.5$), ϵ_t vanishes. Importantly, arranging
92 these tubes within a plane can induce out-of-plane deformation, which can be precisely controlled by adjusting the
93 orientation of the tubes [54–58].

94 *Anisotropic foams.* — Anisotropic foams consist of a network of plates, shells or beams that are preferentially
95 oriented in one direction to impart directional properties (Fig. 1d) [59–61]. By designing foams with spatially varying
96 mechanical properties, one can control how the foam deforms when subjected to external forces or stimuli [61–63].
97 This ability opens up new possibilities for creating materials that can change shape in specific, controlled ways.

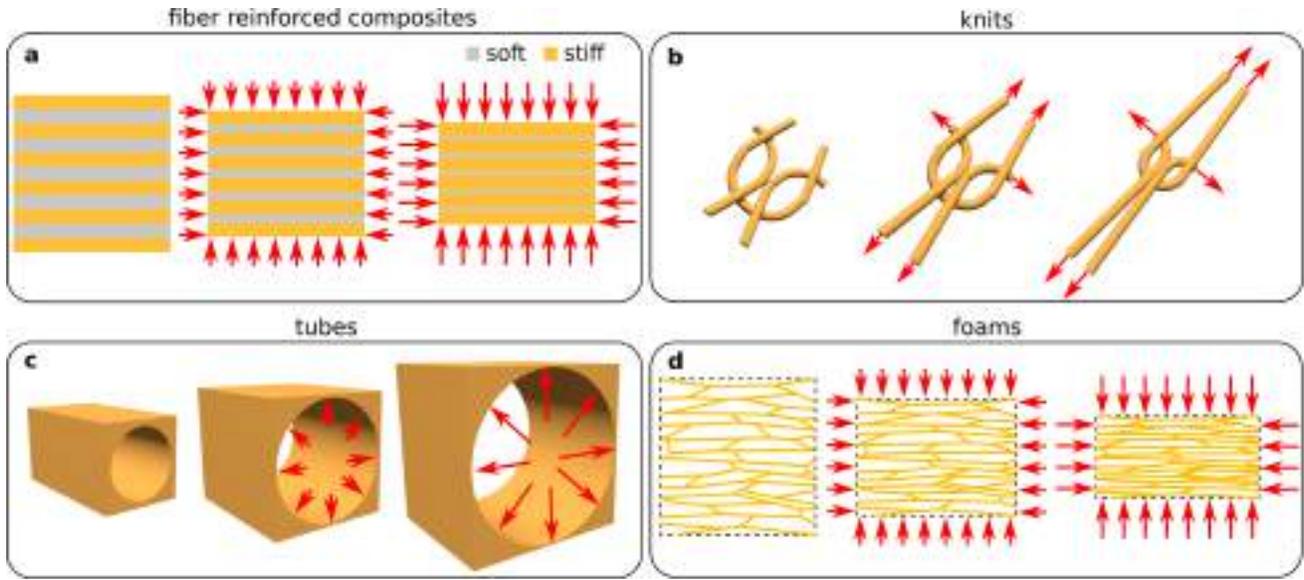


Figure 1. Unit cells based on anisotropic mechanism. Panels show anisotropic unit cells made from **a** fiber-reinforced composites, **b** knits (adapted from [42]), **c** pneumatic tubes and **d** foams that can be used to assemble larger anisotropy-based metamaterials. Red arrows on all panels indicate the applied loading.

98 2.2 Unit cells based on internal rotations

99 Inspired by Ron Resch’s early discoveries, which demonstrated the potential of using rotational motion to realize
 100 shape-morphing metamaterials [64, 65], unit cells based on internal rotations typically consist of stiffer elements,
 101 such as rods or plates, connected by flexible hinges. These stiffer elements generally behave as nearly rigid, with
 102 deformation primarily localized at the hinges. Consequently, the shape change of these units can be described using
 103 relatively simple kinematic models that account for the connectivity and shape of the units. Nevertheless, these hinges
 104 are implemented in practice using thin and flexible elastic components, whose elasticity can significantly influence
 105 the intended kinematics. Evaluating the effect of hinge elasticity on unit cell deformation involves simulating its
 106 behavior. To this aim, discrete models based on a combination of rotational and longitudinal springs to simulate the
 107 hinges’ response have demonstrated high effectiveness [1, 66–71].

108 Unit cells utilizing internal rotations can exhibit either monostable or multistable behaviors. Monostable unit
 109 cells return to their initial configuration once the load is removed and thus require continuous actuation to maintain a
 110 deployed shape. In contrast, multistable unit cells, possess multiple stable configurations, and in turn can be toggled
 111 between various stable shapes [72–82]. Such multistable unit cells exploit the behavior of a von Mises truss—a
 112 structure comprising two rigid rods of length L_{rod} that are free to rotate with respect to each other at one end and are
 113 connected by a spring of stiffness k and rest length L_0 at the other one (Fig. 2a,b). The energy of such von Mises
 114 truss is given by

$$\mathcal{E} = \frac{k}{2}(L_0 - 2L_{\text{rod}} \cos \theta)^2 \quad (3)$$

115 and, as shown in Fig. 2b, it exhibits two minima separated by an energy barrier. The minima corresponds to the two
 116 stable configurations with the truss pointing up or down and the maximum to the truss being flat. This suggests that
 117 one requires two minimal ingredients to achieve bistability for unit cells based on rotating unit cells: (i) a structure
 118 that can switch between two different configurations on either side of a flat state; (ii) some form of confinement. As
 119 we will see below, there are multiple ways of achieving such bistability (Fig. 2c-e).

120 *Rod-based unit cells* One of the best-known designs of rotating rod-based unit cells is that of the re-entrant
 121 hexagon, i.e. a unit cell composed of 8 rigid rods that can rotate relative to each other [85]. Unlike a regular
 122 hexagon, which has all its angles pointing outward, a re-entrant hexagon has some angles that point inward, creating

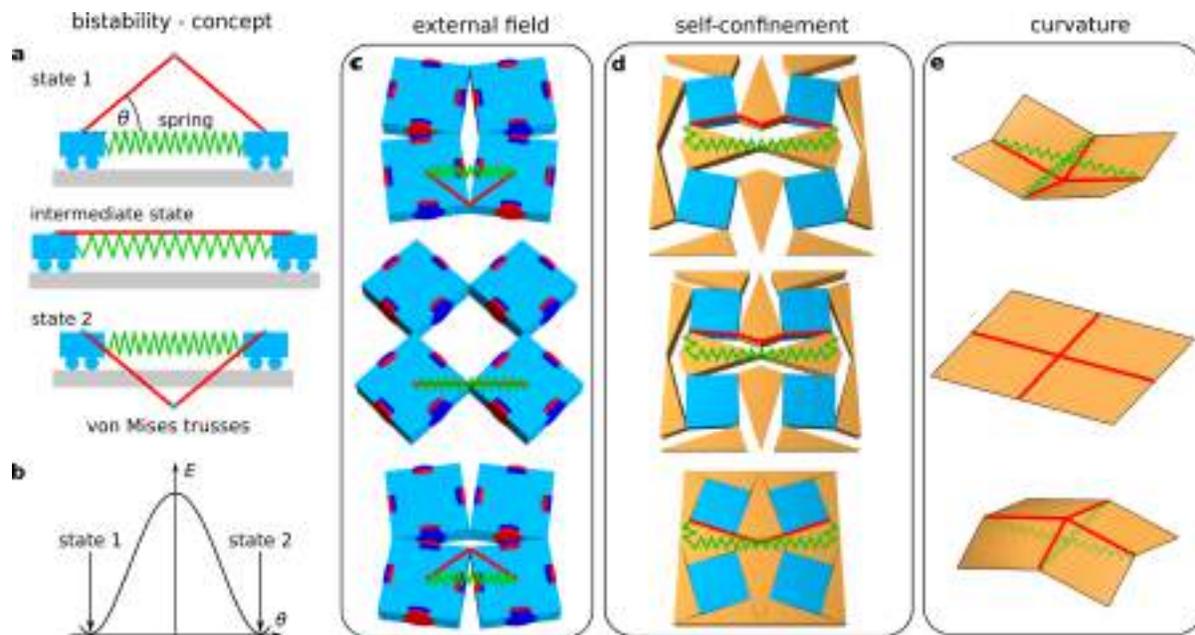


Figure 2. Bistable unit cells. **a** Bistable von Mises truss, made from two inextensible bars that hinge about sliding boundaries (represented by carts). The central spring has a rest length shorter than the added length of the trusses: this makes the flat configuration unstable and instead makes the popped up and popped down configurations stable. **b** Corresponding elastic energy vs. the angle θ . All examples shown in panels **c-e** can be mapped to such a von Mises truss. **c** Magnetic bistable unit cell, where interactions between the rotating units are realized by magnetic interactions (adapted from [83]). The magnets from adjacent squares create an attractive interaction, which makes the unfolded configuration unstable and the two folded configurations stable. **d** Self-confining bistable unit cell, where the unfolding of the unit cell is not kinematically compatible and forces the rotating blocks to stretch (adapted from [84]). **e** Bistable origami unit cell, where self-confinement arises because the vertex has a defect angle (the angles of the plates about the vertex add up to less than 2π). As a result, the plates deform elastically when the vertex is forced through the flat configuration, which is then unstable. The popped-up and down configurations are then both stable, see [5] for an example. In panels **c-e**, auxiliary red lines and green idealized springs are overlaid to represent the inextensible bars and connecting spring of the corresponding von Mises trusses.

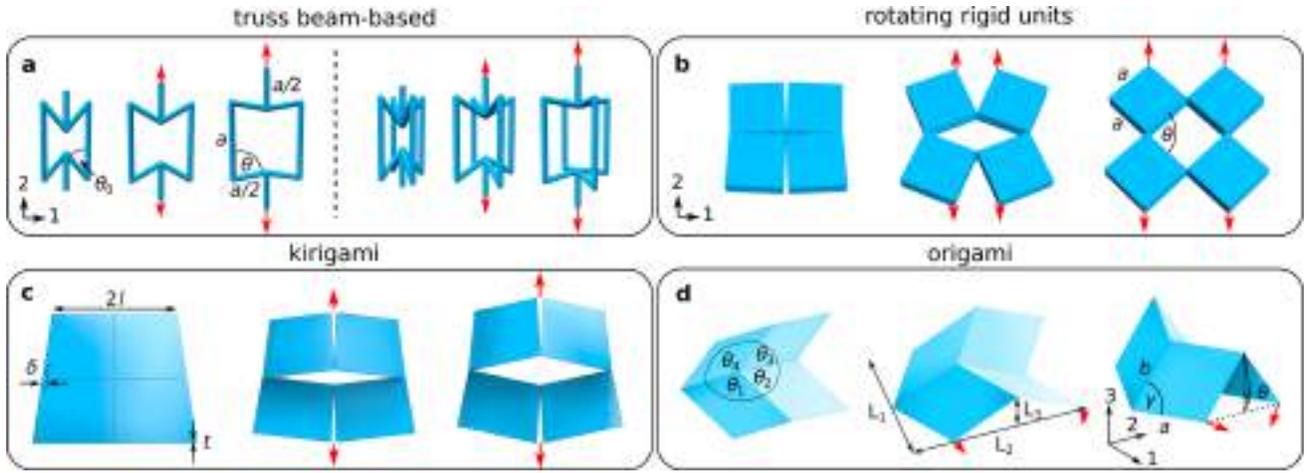


Figure 3. Unit cells based on internal rotations. Panels show unit cells made from **a** rods, **b** rotating blocks, **c** kirigami and **d** origami that can be used to assemble larger rotation-based shape-morphing metamaterials (adapted from [96]). Red arrows on all panels indicate the applied loading.

123 a non-convex shape (see Fig. 3a). In the ideal scenario of re-entrant hexagonal unit cells composed of rigid rods
 124 with lengths a and $a/2$, which are free to rotate at their joints and initially form an angle θ_0 , the application of a
 125 displacement along the direction x_2 changes the angle between the rods to θ . This results in the following expressions
 126 for the strains ϵ_{11} and ϵ_{22} [86]:

$$\epsilon_{11} = \frac{\sin \theta - \sin \theta_0}{\sin \theta_0}, \quad \epsilon_{22} = \frac{\cos \theta_0 - \cos \theta}{2 - \cos \theta_0}. \quad (4)$$

127 It follows that the Poisson's ratio for loading in the direction x_2 is given by

$$\nu_{21} = -\frac{\epsilon_{11}}{\epsilon_{22}} = -\frac{(\sin \theta - \sin \theta_0)(2 - \cos \theta_0)}{(\cos \theta_0 - \cos \theta) \sin \theta_0} \quad (5)$$

128 which is negative for any value of θ and θ_0 in the interval between 0° and 90° . While most materials and structures
 129 exhibit a positive Poisson's ratio, systems with a negative Poisson's ratio (known as auxetic) behave differently: they
 130 contract (expand) in the transverse direction when compressed (stretched). Notably, auxeticity not only enables
 131 unusual shape changes within the plane of the structures but can also be leveraged to control their Gaussian curvature
 132 [87, 88].

133 The re-entrant honeycomb structure is not the only rod-based unit cell that has been suggested. Numerous other
 134 rod-based unit cells with a similar hinging-dominated deformation mechanism have also been proposed [89–95].
 135 The shape-morphing capabilities of rod-based unit cells can be further enhanced by making them bistable and able
 136 to undergo rapid snap-through buckling [77, 78].

137 *Units cell based on rotating blocks* Rigid units connected at their vertices through hinges can rotate with respect
 138 to each other to produce large changes in their covered area. The most famous example of such unit cells is that of
 139 the so-called rotating squares [97, 98], which consist of four rigid squares connected at their vertices through perfect
 140 hinges (see Fig. 3b). When constrained to move in a plane, linear dimensions of such unit cells are given by

$$L_1 = L_2 = 2a \left[\cos \left(\frac{\theta}{2} \right) + \sin \left(\frac{\theta}{2} \right) \right] \quad (6)$$

141 where a denotes the edge length of the squares and θ is the angle between the squares. It follows that this ideal unit
 142 cell exhibits a large negative Poisson's ratio that irrespective of the loading direction is given by

$$\nu = -\frac{\epsilon_{11}}{\epsilon_{22}} = -1. \quad (7)$$

143 Numerous strategies have been proposed to realize this mechanism both in its open (i.e., with $\theta \gg 0$) and closed
 144 (i.e., with $\theta \approx 0$) configuration. The open configuration can be fabricated by embedding arrays of holes in elastic
 145 matrices [98, 99] and connecting rigid units through flexible components [100]. The closed configuration can be
 146 manufactured by embedding arrays of cuts into elastic sheets [101]. In the former, compression is applied to achieve
 147 auxetic behavior, typically triggered by instability [98], while in the latter, auxeticity occurs under tensile loading.
 148 Regardless of the fabrication method, none of the unit cells achieve the ideal Poisson's ratio of -1, as the elasticity of
 149 the hinges limits their rotations.

150 In addition to the rotating squares, a variety of unit cells based on hinged rigid 2D polygons [102–108] and 3D
 151 polyhedra [109, 110] have been proposed. However, for these more general structures, Poisson's ratios are typically
 152 not constant and depend both on the direction of loading and the amount of applied deformation.

153 While unit cells based on rotating blocks are usually monostable, they can also be made multistable [83, 111,
 154 112]. In particular, bistable designs of rotating squares structures have been realized by adding elastic springs [111,
 155 113] or permanent magnets (Fig. 2c) [83, 114, 115], to give an energy cost to the lateral expansion of pairs of squares
 156 as θ decreases, effectively forming a von Mises truss. Furthermore, it has been demonstrated that the confinement
 157 provided by the surrounding elements (Fig. 2d) can effectively lead to the formation of bistable von Mises-like truss
 158 mechanisms within the unit cells [105, 116].

159 *Kirigami unit cells* Kirigami unit cells are created by introducing arrays of cuts into elastic sheets, effectively
 160 forming hinges where rotations localize, ultimately leading to significant changes in shape [117–121]. When the
 161 elastic sheet is thick, kirigami unit cells deform planarly and are equivalent to unit cells based on rotating units
 162 realized in their closed configuration. However, as the thickness t of the sheet decreases, tensile loading may induce
 163 out-of-plane buckling of the hinges, leading to the formation of complex 3D patterns [101, 122, 123] at a critical
 164 strain [101, 124]

$$\varepsilon_{cr} \propto \left(\frac{t}{\delta}\right)^2 \quad (8)$$

165 where δ represents the width of the hinges across the plane. The simplest thin kirigami unit cell consists of a series
 166 of parallel cuts [125]. However, numerous other patterns have been proposed to achieve complex reconfigurations
 167 [123, 126–128], including that of the rotating squares (see Fig. 3c) [101, 129].

168 Finally, kirigami unit cells can be made multistable [130–132]. Most notably, for the standard kirigami pattern
 169 created via a series of parallel cuts [130], the bistable behavior of unit cells originates from the possibility of adjacent
 170 square-like panels to bend out-of-plane to assume stable symmetric or antisymmetric configurations. However,
 171 kirigami created by parallel cuts can also adopt more complex configurations [131–133]. In such scenarios, it is
 172 possible to observe even four stable configurations as a result of the appropriately distributed parallel cuts.

173 *Origami unit cells* In rigid foldable origami, the panels (or facets) between the creases remain rigid and do not
 174 deform. Folding and unfolding occur solely through rotation around the crease lines (hinges), which act as pivot
 175 points. In contrast, folding of a non-rigid foldable origami (also known as flexible or compliant origami) involves
 176 not only the rotation of the panels around the crease lines but also their bending and stretching. The Miura-ori [134]
 177 and the waterbomb [135] patterns are among the most well-known examples of rigid foldable origami metamaterials.
 178 In contrast, non-rigid foldable origami structures include square-twist origami [136–138] and Kresling kirigami
 179 [139, 140]. However, with specific geometric parameters, these non-rigid designs can also be engineered to exhibit
 180 rigid foldability [141].

181 For rigid origami, the configurations induced by folding can be determined by simply applying trigonometry. For
 182 example, for the Miura-ori pattern [88, 96], the linear dimensions of the unit cell can be expressed as (see Fig. 3d):

$$L_1 = \frac{2b \cos(\theta) \tan(\gamma)}{\sqrt{1 + \cos^2(\theta) + \tan^2(\gamma)}}; \quad L_2 = 2a \sqrt{1 - \sin^2(\theta) \sin^2(\gamma)} + \frac{b}{\sqrt{1 + \cos^2(\theta) \tan^2(\gamma)}}; \quad L_3 = a \sin(\theta) \sin(\gamma).$$

183 where, a , b , and γ are side lengths and the internal acute angle of the rigid facets. Furthermore, $\theta \in [0, 90^\circ]$ is the
 184 angle between the facets and the $x_1 - x_2$ plane. On the other hand, for the non-rigid foldable origami, the deformation
 185 process is more complex and typically requires the use of the nonlinear model to describe it [142].

186 Another important classification of origami unit cells is the distinction between flat-foldable and non-flat-foldable
187 designs [143]. Flat-foldable origami, such as the Miura-ori pattern [96, 134], can be completely folded into a flat,
188 two-dimensional shape. This characteristic is particularly useful in applications where compact storage or transport
189 is necessary, and the structure needs to be deployed or expanded when in use. In contrast, non-flat foldable origami,
190 such as the axisymmetric variant of the Miura origami [144], does not fold completely flat but instead transforms
191 into three-dimensional shapes with various functionalities. Researchers have attempted to determine the conditions
192 under which flat-foldability of an origami unit cell can be achieved [145]. Generally, this task is very challenging,
193 but such conditions can be determined for some specific origami patterns such as the vertex-4 pattern [96, 146, 147].

194 Origami unit cells can also exhibit bistable behavior. Similar to rotating rigid block metamaterials, bistable rigid
195 foldable origami can be achieved by exploiting magnetic interactions between strategically distributed magnetic
196 inclusions [148]. More intriguingly, the deformation of faces and hinges that occurs in non-rigid foldable origami
197 can be harnessed to achieve bistability. In non-rigid foldable origami, the faces typically bend to accommodate the
198 folding motion [4, 136]. As a result of face bending in these cases, the structures form von Mises trusses, where they
199 may need to overcome an energy maximum (the elastic energy from face bending) and can thus become bistable.
200 Alternatively, the stretching and shearing of the hinges can be exploited to design origami with two stable states
201 separated by an energy barrier [73, 149]. Another approach is non-Euclidean origami [150, 151], where the sector
202 angles around each vertex sum to $2\pi + \varepsilon$ instead of 2π . An excess angle $\varepsilon > 0$ results in a saddle shape, while a
203 deficit angle $\varepsilon < 0$ leads to a cone shape (see Fig. 2e). This non-Euclidean property makes the flat state energetically
204 unstable, enabling multiple folded stable configurations. This approach has been used to achieve bistable [5, 135,
205 136, 152, 153] and even tristable behavior [150].

206 **3 Different strategies of assembling unit-cells into shape-morphing metamaterials**

207 Mechanical metamaterials capable of shape-morphing are designed by tessellating the unit cells described in Section
208 2. To achieve precise, on-demand shape changes, both the geometry of the individual unit cells, as well as their
209 spatial arrangement, are critical factors. As we will see, shape-morphing can be either monostable—requiring
210 continuous actuation to maintain the shape—or multistable, where the shape is retained even after actuation is turned
211 off. This behavior depends not only on the intrinsic multistability of the unit cells but can also emerge from the
212 collective interaction between them.

213 To provide a unified framework for designing assemblies of unit cells, we propose a classification based on
214 two criteria: (i) whether the deformations of neighboring unit cells are kinematically compatible or incompatible
215 (i.e., whether they are geometrically compatible or frustrated), and (ii) whether the assembly consists of a single
216 periodically tiled unit cell or a combination of various unit cells in the tiling.

217 **3.1 Mechanical metamaterial consisting of compatible unit-cells**

218 We begin by examining metamaterials composed of unit cells that exhibit collective deformations, where each unit
219 cell deforms according to its low-energy mode in a kinematically compatible manner. In this scenario, as previously
220 mentioned, there are two possible sub-cases: periodic and non-periodic arrangements of structural units.

221 **3.1.1 Periodic tiling**

222 The simplest and most widespread way to create a metamaterial is to periodically tile space with a unit cell. For
223 example, take any of the unit cells in Fig. 1 and Fig. 2, tile them in a 2D or 3D lattice and you obtain a metamaterial
224 that will deform following the deformations of the unit cells if the loading is homogeneous and is kinematically
225 compatible with the mode of deformation. Those deformations are typically large as a result of the compliance of the
226 unit cells and can lead to either pronounced shear deformations or volume changes. However, irrespective of the type
227 of reconfiguration, these deformations are homogeneous at the level of the entire metamaterial and therefore lead to
228 limited shape-changing capabilities. With periodic tilings, one can achieve complex on-demand shape-change by
229 exploiting controlled spatial variations of the strain field via (i) local instabilities, (ii) boundary conditions, or (iii)
230 defects. We briefly discuss these strategies below.

231 *Local instabilities.* — The softening caused by snap-through buckling in the unit cells generates metamaterials
232 that deform through a sequence of events [78, 154] (Fig. 4a) and can be utilized for gradual shape-change applications
233 in soft robotics (Fig.4b)[133, 155]. While most metamaterials exhibiting sequential deformation are constructed
234 using multistable unit cells, it is crucial to understand that multistability is not essential for achieving sequential
235 behavior. The critical factor is softening, which can occur through snap-through buckling even in a monostable
236 structure [155].

237 Furthermore, sequential deformation can also arise in hierarchical metamaterials composed of slender elements
238 with varying critical loads. In these systems, different components buckle in sequence once their critical Euler load
239 is exceeded, leading to a series of shape transformations [156].

240 *Boundary conditions*— Complex on-demand shape-changes can also be triggered by carefully selecting the
241 boundary conditions on the metamaterial. Just as a piece of ordinary hyperelastic material, a metamaterial can take
242 arbitrarily complex shapes when a complex load distribution is applied to it. The main difference with ordinary
243 materials though is that the deformations are large as a result of the high compliance of the metamaterial and that
244 specific modes of deformations are favored that differ from that of ordinary materials. Take for instance the simple
245 bending of a slender object, when bent downwards, a plate of rubber will tend to warp upwards on the side as a result
246 of its incompressibility, thus displaying negative Gaussian curvature. In contrast, an auxetic metamaterial will warp
247 downwards because of its auxetic nature, thus displaying positive Gaussian curvature [87, 88]. In fact, as a result of
248 their enhanced compressibility, auxetic metamaterials can more readily change their Gaussian curvature and drape
249 complex surfaces with positive Gaussian curvature, whereas non-auxetic metamaterials can drape complex surfaces
250 with negative Gaussian curvature (Fig. 4c). In addition, deformations of auxetic metamaterials typically consist
251 of dilations, whereas gradients of dilation and shear deformations can be neglected. As a result, theoretical tools
252 such as conformal maps can be used to predict and reverse their shape-changes [69] (Fig. 4d). Hence, especially
253 metamaterials composed of rotation-based unit cells are very sensitive to the application of the nonuniform force
254 and can undergo complex conformal transformations [126, 157]. More generally, metamaterials can be designed to
255 exhibit dominant modes of deformation with enhanced compliance, deformation range, and anisotropy, therefore
256 they can exhibit a wide palette of large and target deformations under complex loading [158].

257 *Defects that create a strain field.* — A third approach involves strategically introducing defects into rotation-
258 based unit cells. These defects create a strain gradient around them, which can be used to induce either in-plane [4,
259 159] or out-of-plane [74, 160, 161] shape changes. Note that the defects can be rigid objects that are initially seeded
260 within the structure [159] (Fig. 4e), or they can be bistable unit cells that can be "popped through" on demand [4,
261 160, 161] (Fig. 4f). Additionally, defects can even spontaneously emerge when a simple uniaxial load is applied to
262 the metamaterial [74] (Fig. 4g). In this latter case, the defects can move and nucleate in sequence, enabling a wide
263 variety of shapes to be achieved with simple loading. However, predicting the strain gradient in the regime of large
264 deformations is computationally expensive, and it is therefore difficult to design for these strain gradients, even with
265 data driven methods (see section 4 for a discussion on design tools). An alternative approach is to directly seed in
266 strain gradients by using non-periodic tilings.

267 **3.1.2 Non-periodic tiling**

268 As previously discussed, periodic tilings can result in complex shape-morphing, but predicting their behavior
269 is challenging due to nonlinear effects such as instabilities. To better design shape-morphing metamaterials,
270 one approach is to assemble dissimilar unit cells. The idea that combining structural components with different
271 mechanical responses can produce interesting shape changes has been recognized for a long time. A classic example
272 is the bimetallic strip [162], which consists of two metal strips that expand at different rates when heated. When
273 joined, these differing expansions cause the flat strip to bend in one direction when heated and in the opposite
274 direction when cooled. This concept can be extended beyond slender structures to the realm of metamaterials. There
275 are several ways to implement this idea. One approach is to spatially grade the unit cell geometry, such as by
276 gradually changing the lattice spacing or unit cell geometry across the material. Another approach is to eliminate
277 periodicity entirely and design fully disordered architectures. A third approach is to maintain some degree of order
278 by designing lattices with consistent periodicity but combining unit cells with different topologies or orientations in

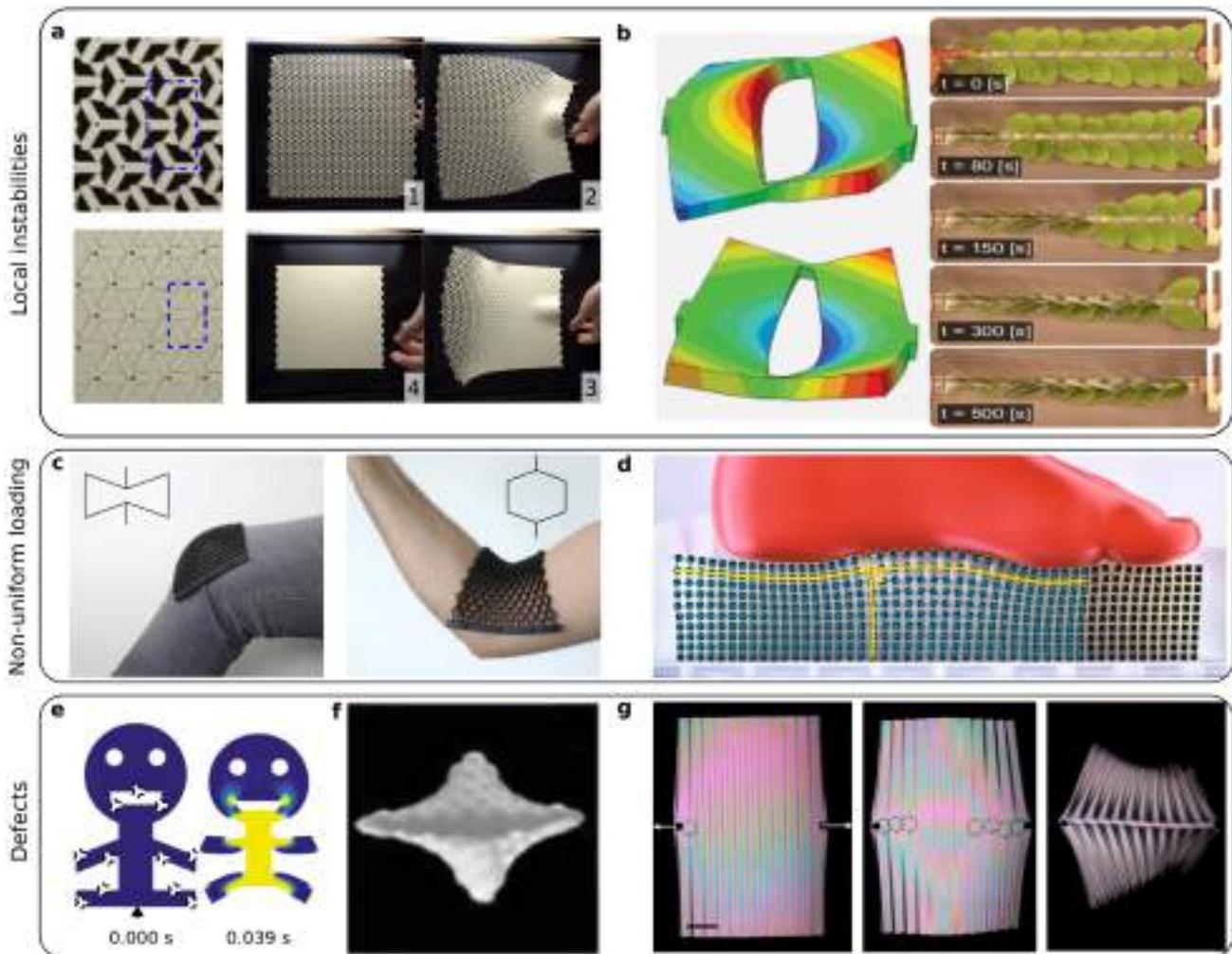


Figure 4. Kinematically compatible metamaterials designed through periodic tiling of structural units. a-b Examples of periodic tiling allowing to achieve complex on-demand shape-change by exploiting local instabilities. **a** In-plane kirigami with a bistable unit cell exhibits an inertial transition wave that takes it from an open to a compact state [159]. **b** Kirigami metamaterial composed of bistable unit cells that sequentially undergo a diffusive transition from the antisymmetric to the symmetric configuration (adapted from [133]). The kirigami is dressed with leaves and mimics the sequential diffusive folding of the mimosa pudica plant. **c-d** Examples of periodic tiling leading to complex on-demand shape-change by exploiting boundary conditions. **c** Metamaterials drape surfaces with positive or negative gaussian curvature depending on their Poisson's ratio (adapted from [87]). **d** Auxetic metamaterials under non-homogeneous loading accomodate conformal deformations (no shear, only dilation and gradients thereof) (reproduced from [69]). **e-g** Examples of periodic tiling allowing to achieve complex on-demand shape-change by exploiting defects that create a strain field around them. **e** Strategic placement of defects enables control over the shape changes (reproduced from [159]). **f** Complex shapes in dome-patterned sheets are created by inverting selected domes (reproduced from [160]). **g** Pulling on a corrugated sheet creates collections of mobile defects that lead to a zoo of shapes (reproduced from [74]).

279 a way that creates an aperiodic or quasi-crystalline structure. These three approaches are discussed below.

280 *Gradients.* — Tiling of spatially varying unit cells is expected to produce spatially varying shape changes, which
 281 collectively generate non-homogeneous strains and result in overall shape transformations at the structural level. For
 282 instance, controlling the arrangement of fibers within a soft matrix has been shown to achieve complex bending

283 and twisting deformations in a thin sheet (Fig. 5a) [33, 35, 163]. However, this concept is not limited to slender
284 structures and can also be applied to systems deforming in-plane. For example, different distortions of the rotating
285 square mechanism from Fig. 1b can be combined, ensuring that each unit cell still fully deploys [164] (Fig. 5b).
286 This enables varying deformations across the metamaterial, allowing for more pronounced deformation inside the
287 structure than outside, which can be used to design a metamaterial capable of deploying from a square to a disk or
288 adopting a specific 3D shape [164]. This approach can be applied to a variety of rotation-based unit cells, including
289 origami [165], kirigami [166, 167], cellular metamaterials [168–170], and anisotropic unit cells, such as inflatable
290 tubes [56] (Fig. 5c).

291 *Disorder.* — Metamaterials with disordered geometries, lacking positional order, can also exhibit shape-morphing
292 capabilities [171, 172]. One example is disordered networks engineered to achieve a specific input/output relationship,
293 a phenomenon known as allosteric [173–175] (Fig. 5e). Another example is anisotropic foams, composed of random
294 microstructures without a repetitive unit cell (Fig. 5f). These foams can be designed using computer graphics
295 techniques such as procedural textures to display an effective elastic tensor with a homogenized anisotropic response,
296 enabling targeted shape changes [62].

297 *Combinatorics.* — A third approach involves arranging structural units with different types or orientations on
298 a periodic lattice to achieve controlled texture morphing [176, 177] (Fig. 5g). As there is only a finite number of
299 possible configurations, such problems have an inherently discrete design space, in contrast with the aforementioned
300 classes that had continuous design spaces. In addition, minute changes in the type of orientation of a single unit cell
301 can completely change the response, e.g. make it switch from floppy to rigid [106, 176, 178, 179] or change the
302 number of low energy deformation modes [106, 179–181] (Fig. 5h). This sensitivity to minute changes renders the
303 problem combinatorial in nature, where only a tiny subset of the design space leads to successful configurations with
304 an on-demand shape-change. The interplay between shape-change and combinatorics can find analogues in other
305 fields of science such as protein folding [182, 183], self-assembly [184, 185], computer graphics [186, 187], and
306 molecular design [188].

307 **3.2 Mechanical metamaterials with kinematic incompatibility between structural units**

308 The range of shapes that mechanical metamaterials can achieve can be expanded by arranging structural units in
309 ways that introduce geometric frustration into their interactions. This kinematic incompatibility between unit cells
310 prevents the system from minimizing all elastic interactions simultaneously, often resulting in mechanical instabilities
311 that can induce significant shape changes from small inputs. Since shape-morphing in periodic arrangements of
312 kinematically incompatible structural units is primarily driven by instabilities, designing systems that achieve
313 target shapes becomes more complex and requires a finer understanding of the nonlinear mechanics at play and
314 more advanced optimization tools. Consequently, this approach has been less explored, but it holds intriguing
315 potential: small inputs can trigger large shape changes, and multiple degenerate states can emerge, between which
316 the metamaterial can switch.

317 For metamaterials with kinematic incompatibility, as with compatible metamaterials, we classify them into two
318 categories: those with unit cells organized through periodic tiling and those organized through non-periodic tiling.

319 **3.2.1 Periodic tiling**

320 Geometric frustration in periodic arrangements of unit cells can be introduced by leveraging buckling in loops that
321 impose a global constraint. This phenomenon can be illustrated by examining elastic beams connected to form
322 two-dimensional square and triangular frames [190]. In the unfrustrated square frame, each beam can buckle into its
323 lowest-energy configuration—a half sinusoid—while simultaneously preserving joint angles with neighboring beams,
324 minimizing deformation energy (Fig. 6a). In a triangular frame, however, such configurations are unattainable,
325 resulting in a frustrated system. Notably, it has been shown that buckling in frustrated triangular cellular structures
326 leads to the formation of complex, ordered patterns [190].

327 *Tiling of unit cells based on internal rotations.* — Geometric frustration can be introduced in periodic arrange-
328 ments of rotating squares by creating loops with an odd number of units. In a structure made of rotating squares,
329 the energy is minimized when neighboring units rotate in opposite directions. However, this preferred deformation

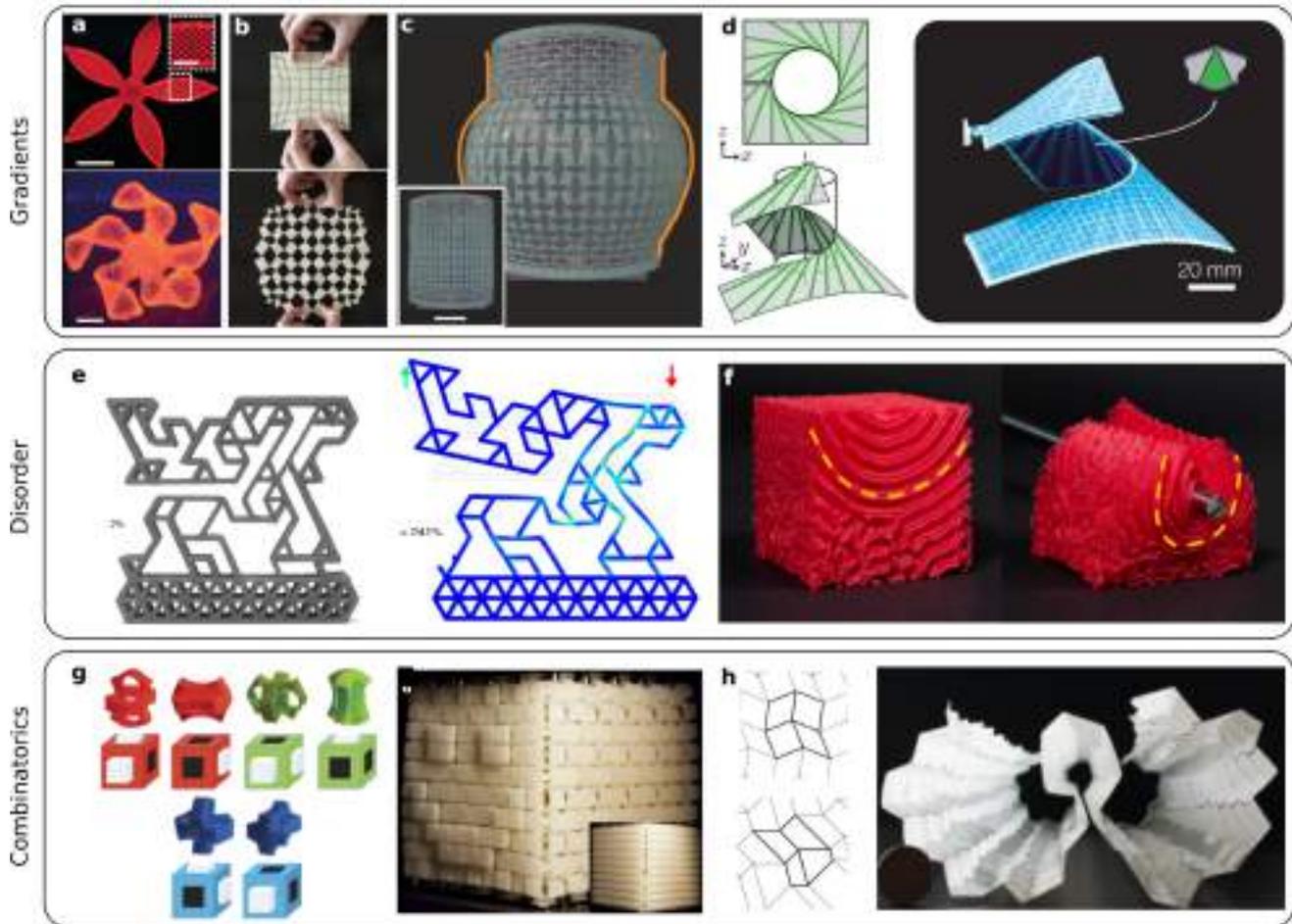


Figure 5. Kinematically compatible metamaterials designed via non-periodic tiling of structural units. a-d. Examples of non-periodic tiling that allow to achieve complex on-demand shape-change by exploiting gradients. **a** Composite hydrogel architecture encoded with localized, anisotropic swelling behaviour controlled by the alignment of cellulose fibrils along prescribed pathways (reproduced from [35]). **b** Kirigami pattern programmed for getting a square to a circle on deployment (reproduced from [164]). **c** Kirigami inflatable that mimic target jar profiles upon pneumatic actuation (reproduced from [166]). **d** Developable helicoid made from trapezoidal channels 3D printed on a layer of airtight fabric (reproduced from [56]). **e-f** Examples of non-periodic tiling allowing to achieve complex on-demand shape-change by exploiting disorder. **e** Disordered architecture programmed to achieve a target movement upon mechanical loading (reproduced from [175]). **f** Disordered foam with fiber oriented to control how the volume reshapes under large deformations (reproduced from [62]). **g-h** Examples of non-periodic tiling that allow to achieve complex on-demand shape-change by exploiting combinatorics. **g** 3D cubic metamaterial made from aperiodically oriented building blocks reveals its precisely designed surface texture under uniaxial compression (reproduced from [189]). **h** Combinatorial origami made from a discrete set of origami vertices can deploy into multiple target shapes (reproduced from [180]).

330 mode cannot be supported in cylinders [191, 192], rings, or tori [70] with loops containing an odd number of squares.
 331 Along these loops, local constraints cannot all be satisfied simultaneously, as not all neighboring square pairs can
 332 rotate in opposite direction. This incompatibility between the optimal deformation mode of the rotating squares
 333 and the odd number of units in the loops makes the system geometrically frustrated, resulting in a topologically
 334 protected line of defects that can be easily moved with small forces (Fig. 6b). This feature enables the creation of a
 335 robust programmable memory [70], but how to harness such topological frustration for on-demand shape changes

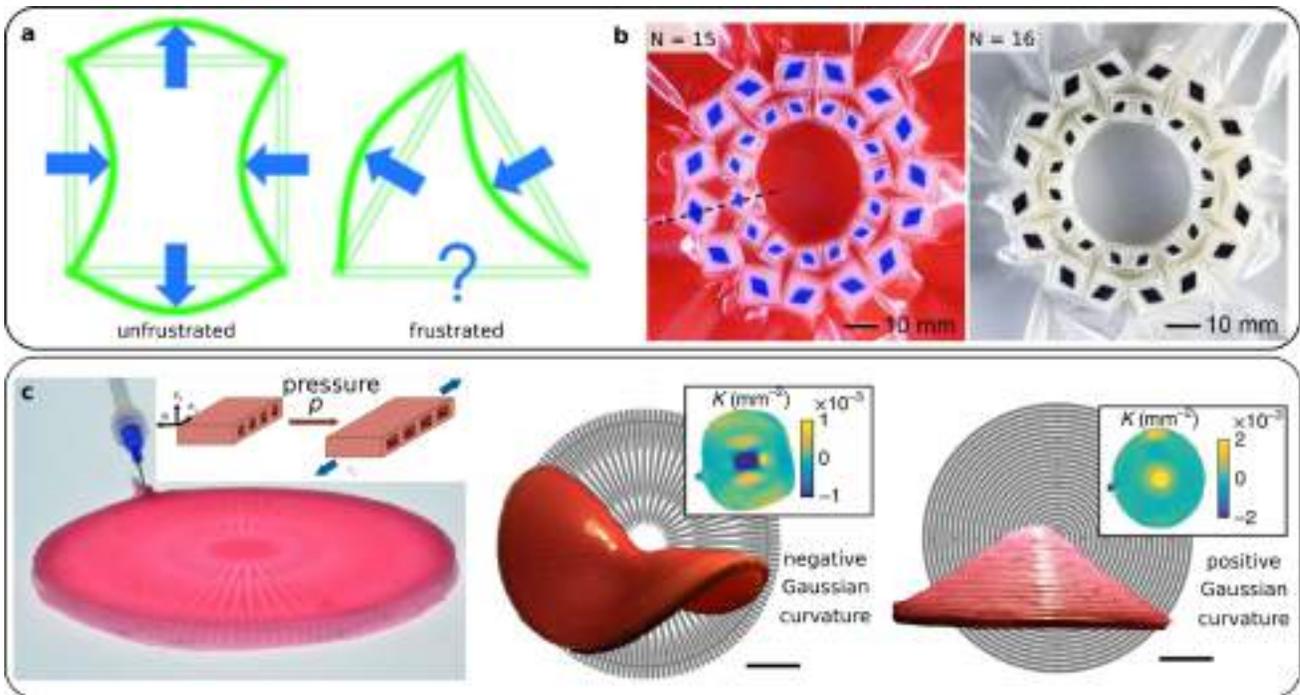


Figure 6. Kinematically incompatible metamaterials designed via periodic tiling of structural units. a-b Geometric frustration from loops with an odd number of units. **a** Buckled beams on frames want to preserve angles at joints to minimize the deformation energy. This can be realized for square frames, but not for triangular frames which become geometrically frustrated (reproduced from [190]). **b** This concept can be generalized to metamaterials made from global loops, where the parity of the loop defines whether the metamaterial is frustrated or not (reproduced from [70]). **c** Frustrated plate-shaped metamaterial made of anisotropic tubes (reproduced from [54]).

336 with memory remains an intriguing open question.

337 *Tiling of unit cells based on anisotropic mechanism.* — A global constraint can also be readily created by
 338 arranging tubes and filaments into circular plates. A notable example is that of anisotropic tubes arranged to form a
 339 circular plate [54] (Fig. 6c). If the tubes deformed equally in all directions, the plate would remain flat. However,
 340 upon pressurization, the tubes elongate more in the circumferential direction than along the longitudinal one (see
 341 Equation (2)). This anisotropic deformation cannot be accommodated by a flat plate, causing it to buckle out of
 342 plane to reach a 3D equilibrium shape that minimizes total elastic energy. For example, if the tubes are arranged
 343 radially, the plate expands more in the azimuthal direction than in the radial direction upon pressurization. This
 344 creates an excess angle in the plate, which destabilizes it into a shape with negative Gaussian curvature (Fig. 6c).
 345 Conversely, if the tubes are arranged in concentric circles, the plate expands more in the radial direction, triggering
 346 an instability that results in a conical shape (Fig. 6c). Note that similar shape changes have also been observed in
 347 anisotropic plates comprising a dielectric elastomeric matrix reinforced with stiff fibers [193].

348 **3.2.2 Non-periodic tiling**

349 Periodic arrangements of kinematically incompatible structural units can only exhibit geometric frustration in the
 350 presence of loops. However, this constraint is unnecessary when using non-periodic tiling, as frustration can also
 351 arise from mismatched strains across different regions within the structure. Early examples of this concept were
 352 demonstrated in polymer gel plates patterned with materials exhibiting varying swelling ratios [194] (Fig. 7a).
 353 Differential swelling creates in-plane strain variations across the plate, leading to kinematic incompatibilities that
 354 drive changes in Gaussian curvature and trigger out-of-plane buckling instabilities.

355 This approach has enabled complex shape changes in metamaterials made from both rotating and anisotropic
 356 unit cells. Metamaterials corresponding to plates with spatially varying dimensions have been designed to induce

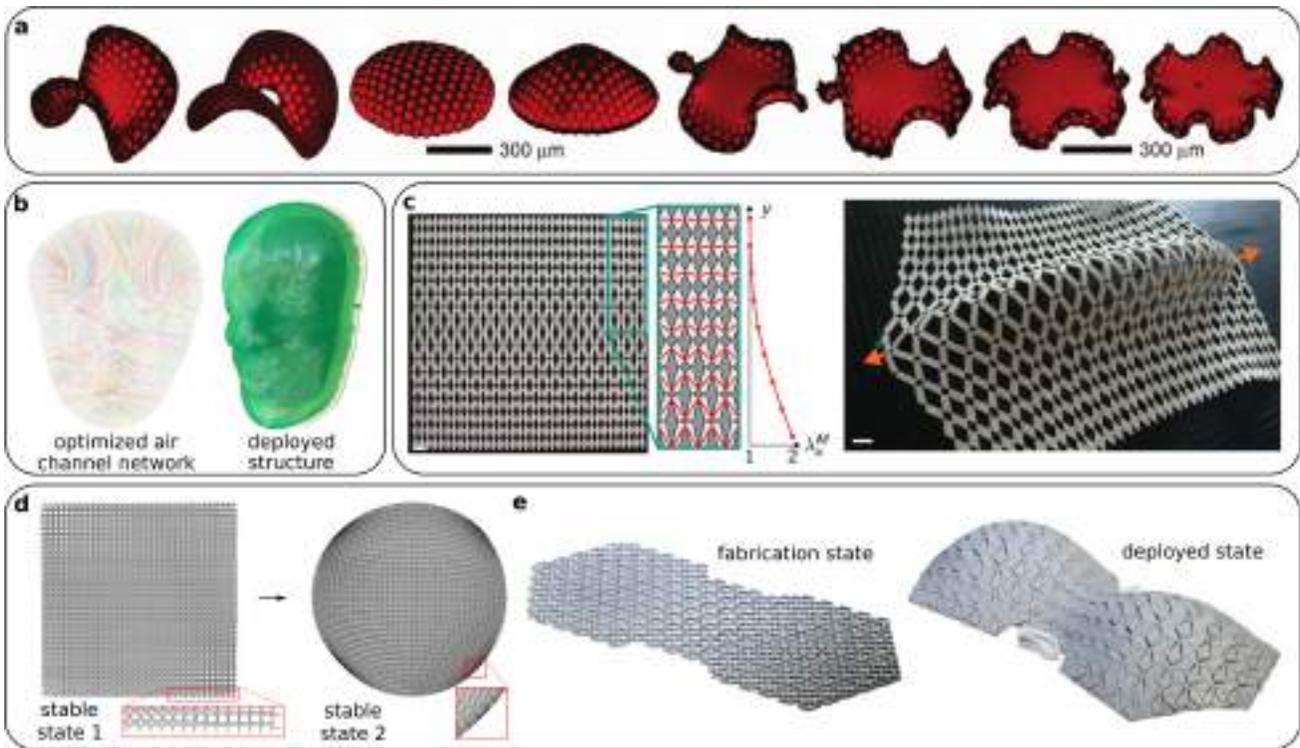


Figure 7. Kinematically incompatible metamaterials made from non-periodic arrangements of structural units. **a** Polymer gel plates patterned with varying swelling ratios buckle out of the plane in complex shapes that depend on the pattern (reproduced from [194]). **b** Inflatible plate made from non-periodic arrangements of pneumatic tubes designed by computational optimization (reproduced from [57]). The plate buckles out of the plane into the shape of the human face. **c** Metamaterial composed of quads connected by their vertices and of spatially varying dimensions (reproduced from [195]). The quads counter-rotate under stretch. The tiling expands to various degrees of strain across the metamaterial and hence leads to out-of-plane buckling under stretch. **d-e** Metamaterials made from rotating units deforming either **d** in-plane (reproduced from [112]) or **e** out-of-plane (reproduced from [196]) as a result of the kinematic incompatibility of their deformations. This kinematic incompatibility leads to an energy maximum the structure needs to deploy. As a result the deployed shapes are stable even when the loading is removed.

357 strain differences across the sample that trigger out-of-plane buckling instabilities and the formation of complex 3D
 358 morphologies [195, 196](Fig. 7c). When the metamaterial is sufficiently thick, however, it deforms in-plane. In this
 359 case, geometric frustration can lead to bistability, where both the initial and deployed shape are stable [112, 164,
 360 197] (Fig. 7d-e).

361 Similarly, anisotropic unit cells with varying orientation and shape can be combined to form plates that experience
 362 spatially varying strains. Such strain variations in the plane force changes in Gaussian curvature, causing the plate to
 363 buckle into a three-dimensional shape [54, 57, 198, 199] (Fig. 7b).

364 4 Design Tools for achieving shape-morphing.

365 When designing shape-morphing metamaterials, the goal is often to create structures that can transform into specific,
 366 predefined target shapes. This design process can be framed as an inverse problem, where we aim to identify a
 367 metamaterial architecture capable of achieving the desired shape through mechanical deformation.

368 Robust and efficient algorithms such as topology optimization [200] have been developed to guide the design of
 369 structures with target responses in the linear regime. However, these methods are not directly applicable to the inverse
 370 design of shape-morphing mechanical metamaterials. This is due to the highly nonlinear nature of these systems,

371 which often exhibit complex energy landscapes with multiple minima separated by large energy barriers, making
372 them challenging to navigate. Consequently, solving the inverse problem for nonlinear metamaterials requires first
373 selecting the design space (i.e. a set of unit cells and assembly rules). Once these are identified, computational tools
374 can then be deployed to optimize the geometry of the unit cells, enabling the desired shape transformation.

375 *Choosing the design space.* — The first step involves defining the design space and constraints. After selecting
376 the type of unit cells and their tessellation pattern, a model must be developed to capture their behavior. There
377 are two main approaches to modeling the response of shape-morphing metamaterials: kinematic constraints or
378 finite energy deformations. In the kinematic approach, the elasticity of the metamaterial is disregarded, assuming it
379 deforms according to an underlying rigid mechanism [164]. Metamaterials with a single underlying mechanism are
380 ideal for robust shape-changing. Differently, mechanisms with multiple degrees of freedom allow for various target
381 shapes, but are less robust and require more complex actuation [201].

382 However, it is important to note that fabricated metamaterials based on rotating units do not typically support
383 idealized, mechanism-like behavior. Fabrication constraints, such as minimum hinge width, result in hinges
384 with notable stiffness that dampens the desired mechanism-like response [202]. In such cases, models relying
385 solely on kinematics are insufficient, and fully elastic models must be employed, though these are generally more
386 computationally demanding. To reduce computation costs, these models often simplify elastic problems by using
387 plate [57, 198, 199], beam [203, 204], or mass-spring [205–207] elements. Alternatively, detailed continuum models
388 can be accelerated by surrogate models with AI [208, 209], but the use of such data-driven models is not very well
389 developed yet in the context of shape-morphing.

390 *Computational design.* — Once the design space has been selected, one can turn the design of a target shape-
391 change into a computational optimization problem. The cost function is the distance between the actual and target
392 shape change and the design parameters are the geometrical parameters of the metamaterial. If one is concerned
393 with zero energy deformations, the optimization has then to be performed under constraints to ensure that the
394 deformations remain at zero energy throughout the optimization [3, 164, 165, 210–213]. If one is concerned with
395 finite energy deformations, one does not require such constraints, but the challenge is to have an efficient forward
396 simulation to reduce computation time.

397 **5 Outlook**

398 In summary, this review has examined the different types of shape-morphing metamaterials developed to date and
399 systematically identified the design strategies used. We conclude by highlighting key challenges for future research.

400 *Load-carrying capacity.* An obvious challenge is to create shape-morphing structures that can carry a load.
401 Promising approaches are being introduced that use the concept of tensegrity via stiff fibers in tension [55], granular
402 contacts in compression [214, 215], and the use of stiff materials that display plasticity [216]. Yet systematic design
403 remains hard. An active effort is currently taking place to meet this challenge notably by using multi-objective
404 optimization methods [57, 217] with exciting prospects for architecture.

405 *Multiple shapes.* Most of the shape-morphing metamaterials either can morph into one shape only, or into multiple
406 shapes but at the cost of complex actuation. Despite early attempts using loading speed [133] or multistability [218],
407 creating multishape metamaterials with simple actuation remains a formidable challenge. The use of multistability
408 discussed above throughout this paper is emerging as a particularly promising avenue to create metamaterials that
409 can toggle between different stable shapes. As a result of this multistability, the state the metamaterial sits in
410 depends on the loading history and in some cases many different states can be visited by suitably controlling the
411 loading and unloading sequence. This approach is now routinely employed for mechanical computing [219–222],
412 but remains largely open in the context of metamaterials for shape-morphing [218]. Another related challenge is
413 how to perform the design of such multishape metamaterials. So far attempts to perform computational design of
414 multishape metamaterials are able to create only relative simple shape-morphing that in addition are often muddled
415 by spurious modes [3, 180, 181, 210, 212].

416 *Micron-scale fabrication.* An equally difficult challenge is to scale shape-morphing metamaterials down to the
417 micron-scale. There the key challenge is the fabrication and the actuation. Although two-photon nanolithography can

418 be used to create metamaterials with features as small as 150nm, bulky instruments such as nanoindenters are required
419 to make them shape-change. Another particularly promising approach is to nanofabricate responsive materials such as
420 elastomers that can swell [136], graphene bibeams [223–225] and magnetic domains [226, 227]. Such metamaterials
421 are always made by lithography or atomic layer depositions as two-dimensional origami or kirigami, thus the key
422 challenge is to select the right combination of materials that offer compliance and responsiveness simultaneously and
423 to design the folding out of the plane and the shape-morphing once deployed. Those metamaterials offer compelling
424 potential for applications in microrobotics in aqueous environments [223–225].

425 *Manipulating electromagnetic wave fronts.* Another exciting application of shape morphing metamaterials is the
426 ability to tune on the fly how electromagnetic waves such as radiofrequency waves and light are being steered, filtered,
427 or shaped [11–13, 228–244]. Electromagnetic metamaterials often harness the strong sensitivity of resonators to
428 mold the flow of waves such as light [245–248]. The relative position and orientation of these resonators is crucial to
429 achieve precise control over the intensity and phase of the waves that are scattered off the metamaterials. Through
430 mechanical deformation, it is possible to dynamically change the relative position and orientation of these resonators
431 and hence the electromagnetic functionality. Such tuning is traditionally done through rudimentary mechanical
432 deformations such as stretching membranes [228–231, 243, 249] or deformations out of the plane [250, 251]. Owing
433 to the more complex deformation pathways that they can achieve, shape-morphing metamaterials hold the potential
434 to reach much more advanced dynamically tunable electromagnetic functionalities [10–13]. Just as in the previous
435 section, an outstanding challenge is to miniaturize fabrication and actuation to achieve dynamical control of visible
436 light [226, 227].

437 *Robotics.* A fascinating area of application of shape-morphing metamaterials is that of robotics, for e.g.
438 locomotion or gripping. Yet, for those robotic functionalities, not only does the metamaterial need to shape change,
439 the shape-change is required to occur in a time-ordered fashion or a cycle, in order to do work on its environment.
440 Although recent works have used solitons [111, 252] or limit cycles [253, 254] to achieve locomotion, how do design
441 a cycle of shape changes rather than a unique shape change and how to control its dynamics remains a largely open
442 problem.

443 Authors contributions

444 C.C., K.B., and K.K.D. wrote the first draft of the article. K.K.D., C.C., and M.K. prepared the figures based on the
445 concept proposed by K.B. and C.C. All authors reviewed and/or edited the article before submission and contributed
446 to the discussion.

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