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# Improvement of Silicon Nanotweezers Sensitivity for Mechanical Characterization of Biomolecules Using Closed-Loop Control

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Abstract—In this paper, we show that closed-loop control can be advantageously used for the characterization of mechanical properties of biomolecules using silicon nanotweezers (SNT). SNT have already been used in open-loop mode for the characterization of mechanical properties of DNA molecules. Up to now, such an approach allows the detection of stiffness variations equivalent to about 15 DNA molecules. Here, it is shown that this resolution is inversely proportional to the resonance frequency of the whole system and that real-time feedback control with state observer can drastically improve the performances of the tweezers used as biosensors. Such improvement is experimentally validated in the case of the manipulation of fibronectin molecules. The results are promising for the accurate characterization of biopolymers such as DNA molecules.

*Index Terms*—Biosensor, MEMS tweezers, parameter detection, sensitivity improvement, state feedback.

#### I. INTRODUCTION

RECENT developments in micronano manipulation tools have revealed crucial information on the mechanical behavior of biomolecules [1]–[3]. These manipulations generally performed on a single molecule have given the quantitative data needed to elucidate fundamental biological processes as DNA wrapping [4] and replication [5], or cell cytoskeleton dynamics through actin filament [6], and microtubule mechanical responses [7].

Several methods are already available for performing biological experiments at the molecular level: magnetic tweezers [8], [9], optical tweezers [10], AFM cantilevers [11], and microfibers [2], [12]. A single filamentary macromolecule, such as DNA, is first attached to a surface at one end, and to a bead

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or a flexible cantilever at the other end. Forces or displacements are sensed by optical measurements. Nevertheless, the real-time operation and the routine implementation of these techniques remain difficult to achieve, as they require complex experimental procedures. In this respect, microelectromechanical systems (MEMS) offer an advantage for systematic analysis since accurate molecular level tools (actuator, end effectors, and sensor) can be integrated on a MEMS platform. Furthermore, these devices can be produced at low cost with batch fabrication, as have the microgrippers developed to grab microsized objects [13] or cells [14]. However, up to now, the performances of such tools are not sufficient to deal with single molecule characterization. In previous reports, silicon nanotweezers (SNT) were designed and fabricated to trap DNA molecules [15], [16] and characterize DNA bundles [17] and reactions on DNA [18]. The stiffness of DNA bundles was derived from the open-loop measurement of the mechanical resonance frequency shift. The current sensing resolution is in the range of  $1 \text{ mN} \cdot \text{m}^{-1}$ , which corresponds to approximatively 15  $\lambda$ -DNA molecules [20]. This resolution is limited by the design of the tweezers and the noise of the electronic instrumentation. To improve the overall system sensitivity towards lower stiffness, the primary route consists of reducing the tweezers stiffness to bring the sensor and measured object characteristics close to the same range. Unfortunately, such a design would result in a system that would be too fragile to be processed and utilized.

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Our approach consists of scaling down the resonance frequency of the closed-loop system to enhance the sensitivity to SNT stiffness change. After a proper identification of the SNT model, the feedback control has been simulated, optimized, and then implemented. This control strategy has been evaluated with thin *fibronectin* molecular bundles trapped between SNT tips, and we propose to discuss the advantages and the limits of such closed-loop operations.

This paper is organized as follows. Section II describes the tweezers, the sensing principle, and provides an example of monitoring the DNA trapping. The dynamic model of the tweezers is then detailed in Section III-A. The closed-loop control strategy and its potential to improve the system sensitivity are discussed in Sections III-B and III-C. Section IV details the experimental implementation and provides information on *fibronectin* molecules measurement with different

 $<sup>^1\</sup>lambda\text{-DNA}$  is DNA of the bacteriophage  $\lambda$  having a contour length of  $16.4~\mu m$  and a stiffness in elastic regime of  $66~\mu N \cdot m^{-1}$  in low ionic solution [19].

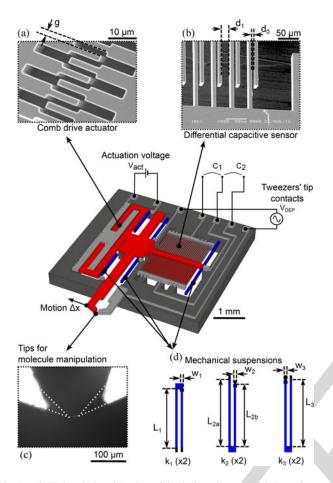


Fig. 1. SNT description. 3-D view of the device whose overall dimensions are  $4.5 \times 4.5 \times 0.5 \,\mathrm{mm}$ . The mobile parts are shown in red and the mechanical suspensions in blue. (a) Scanning electronic microscopic (SEM) image of the actuator consisting of 880 pairs of interdigitated combs separated by a 2- $\mu\mathrm{m}$  gap, and organized in two series. (b) SEM image of the integrated sensor of two opposing series of 30 combs. (c) Microscope image of the tweezers tips with a DNA bundle in between. (d) Design of the three sets of suspensions,  $k_1$  for the actuator polarization,  $k_2$  for the mobile arm, and  $k_3$  for central sensor plate.

closed-loop settings. Closing remarks and perspectives are given in Section V.

## II. DESCRIPTION OF THE DEVICE AND ITS USE FOR THE MECHANICAL CHARACTERIZATION OF BIOMOLECULES BUNDLES

#### A. Device Description

- 1) Design: Fig. 1 shows a three-dimensional illustration of the SNT device. It consists of two arms terminated in sharp tips that act as electrodes to trap biomolecules by dielectrophoresis (DEP) forces [15]. One tip is fixed; the other one is moved by an electrostatic actuator. The motion of the mobile electrode is measured by capacitances whose value linearly varies with electrode displacement; two variable capacitances are mounted in differential mode [21].
- 2) Electrostatic Actuation: The mobile arm is actuated by attractive electrostatic forces generated in an interdigitated comb drive [see Fig. 1(a)], one of the most widely used architecture

in MEMS design [22], [23]. The maximal stroke is limited compared to that achievable by a parallel plate actuator but the generated force only depends on the actuation voltage and not on the electrode position, thus, enabling a simpler actuation control. The force is given by (1) where  $\varepsilon_0$  is the vacuum permittivity (we consider the relative permittivity of air = 1),  $N_a$  is the number of comb fingers, t is the device thickness, t is the lateral gap between fingers, and t is the actuation voltage. Due to its capacitive structure, the comb drive generates attractive forces between its two electrodes

$$F_{\rm es} = \frac{1}{2} \frac{\varepsilon_0 \times N_a \times t}{g} V_{\rm act}^2 = \alpha_{\rm es} \times V_{\rm act}^2.$$
 (1)

3) Mechanical Structure: The mobile part of the tweezers is linked to the silicon frame by flexible beams [see Fig. 1(d)]. Commonly integrated in mechanical microsystems [23], [24], folded beam springs are designed to minimize beam areas, decrease their mechanical stiffness and enhance displacement ranges [25]. In the SNT design of Fig. 1, three sets of folded beam suspensions support the mobile part of the system (the comb-drive actuator, the mobile tip, and the capacitive sensor) and provide the electrical connections for the actuation and the sensing. The springs are arranged symmetrically along the actuation and sensing axis to minimize any rotation, and their sum gives the total stiffness k of the device.

A highly compliant system is required to sense the mechanical characteristic of trapped molecules on the tweezers response. On the other hand, a minimum stiffness is mandatory, 1) to endure the fabrication processes and manipulations, 2) to support the mobile system weight, and 3) to prevent sticking due to attractive surface forces between the comb-drive actuator and the capacitive sensor electrodes.

4) Displacement/Velocity Sensing: The tip position is measured by a capacitive sensor designed in a triplate configuration with transverse combs [see Fig. 1(b)] [26]. The central electrode, linked to the mobile arm, moves in between two fixed electrodes creating the two differential capacitances  $C_1$  and  $C_2$  whose difference,  $\Delta C$ , is related to the displacement x. For small displacements (i.e., x is much smaller than the gaps between electrodes  $d_0$  and  $d_1$ ),  $\Delta C$  is proportional to x [17],  $N_b$  is the number of capacitance electrodes, and L the length of the electrodes

$$\Delta C = C_1 - C_2 \tag{2}$$

$$\simeq 2\varepsilon_0 N_b Lt \left(\frac{1}{d_0^2} - \frac{1}{d_1^2}\right) x = \beta_{\rm C} \times x$$
 (3)

5) Electronic Read-Out: The tweezers motion is sensed by the measurement of the differential capacitance  $\Delta C$  (2) through current sensing. In dynamic mode, a dc voltage ( $V_{\rm polar}$ ) is applied on the mobile central electrode, whose motion generates dynamic currents  $i_1$  and  $i_2$  flowing through the capacitances  $C_1$  and  $C_2$ , respectively, [27], [28]. The resulting currents related to the motion velocity are converted into voltages  $V_1$  and  $V_2$  by two low-noise current-to-voltage (A/V) preamplifiers (Signal Recovery, model 5182). The low input impedance of the preamplifier (virtual ground) ensures an accurate current conversion [17], [29].

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TABLE I NUMERICAL VALUES OF DIMENSIONS AND PARAMETERS OF THE SNT

Silicon thickness	
t (μm)	30
Comb drive actuator	
$N_a$	440
$g (\mu m)$	2
Mechanical suspensions	
$L_1, L_{2a} (\mu m)$	900
$L_{2b}, L_3 (\mu m)$	1000
$w_1, w_2, w_3 \; (\mu \text{m})$	15
Capacitive sensor	
$N_b$	30
$L(\mu m)$	585
$d_0(\mu m)$	5
$d_1 (\mu m)$	20

The device dimensions are defined in Fig. 1.

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Finally, in harmonic mode, when the tweezers is moved by a sinusoidal actuation, a lock-in amplifier (NF, model LI 5640) performs the low-noise detection of the differential signal (amplitude and phase of  $V_1 - V_2$ ) at the actuation frequency. The reference signal is the motion frequency imposed by the actuation voltage.

All dimensions and parameters that has been considered for a proper model of the SNT are summed up in the TableI. Furthermore, Young's modulus and the density of the silicon are, respectively, 165 GPa and 2329 kg  $\cdot$  m<sup>-3</sup>, and the permittivity  $\varepsilon_0$  is  $8.85 \times 10^{-12} \text{ kg}^{-1} \cdot \text{m}^{-3} \cdot \text{A}^2 \cdot \text{s}^4$ .

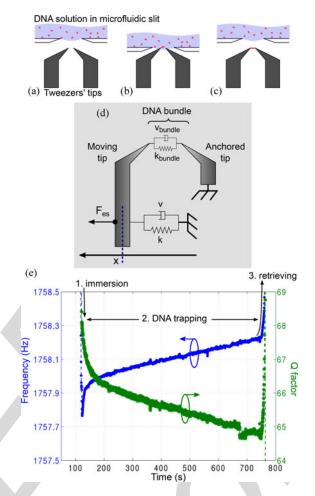
#### B. Monitoring Biomolecules Manipulation With SNT

This Section presents the way that the SNT are used as biosensors and points out the measurement sensitivity requirements. This example is illustrated with DNA but the same technique has been applied to gelatin [30] and microtubules [31].

1) DNA Bundle Trapping by DEP: The trapping is achieved by applying an ac electric field ( $E = 1 \text{ MV} \cdot \text{m}^{-1}$ ) between the two opposing tips of the tweezers [32] once partially immersed in the solution. The retrieving of the DNA bundle is routinely performed from a droplet solution [15], [17], and single molecule trapping has even been demonstrated with a pulsed DEP signal [16].

In [20], an improved method is proposed in which the tips are introduced into an open fluidic cavity optimized to guaranty a stable meniscus. The SNT position is fixed and the cavity is mounted on a XYZ stage controlled by programmable robot for repeatable and controlled tip immersion, as illustrated in Fig. 2(a)–(c).

2) Real-Time Monitoring: The SNT characterization allows monitoring of the DNA trapping in real time. The characterization of the SNT in real time allows the monitoring of the DNA trapping. The number of trapped DNA molecules during the DEP is controlled by the evolution of the resonance frequency and quality factor of the system (Tweezers+DNA bundle). The SNT are modeled by a linear second-order model as depicted in Fig. 2(d). The resonance frequency,  $f_{\rm R}$ , and the quality factor



 $\lambda$ -DNA molecules trapping with SNT. The first three sketches illus-Fig. 2. trate the key phases of the experiment: (a) the immersion of the tips in the DNA solution, (b) the DNA trapping by DEP, and (c) the retrieving of DNA bundle between the tips. (d) Equivalent dynamic model of the SNT with bundle. (e) Evolution of the SNT+bundle resonance frequency and Q factor during the trapping.

Q of the SNT+bundle damped oscillator are defined by

$$f_{\rm R}(t) = \frac{1}{2\pi} \sqrt{\frac{(k + k_{\rm bundle}(t))}{M}}$$

$$Q(t) = \frac{\sqrt{(k + k_{\rm bundle}(t)) \times M}}{(\nu + \nu_{\rm bundle}(t))}$$
(5)

$$Q(t) = \frac{\sqrt{(k + k_{\text{bundle}}(t)) \times M}}{(\nu + \nu_{\text{bundle}}(t))}$$
 (5)

where M is the mass of the movable tip, k is the stiffness of the suspensions, and  $\nu$  is the equivalent viscous losses.  $k_{\rm bundle}$  and  $\nu_{\rm bundle}$  are the time varying bundle stiffness and viscosity; the mass of the bundle is neglected compared to M.

As shown in Fig. 2(e), as the DEP trapping progresses, the system moves to higher resonance frequencies indicating an increase in the bundle stiffness according to (4). At the same time, the signal amplitude at resonance, proportional to the quality factor Q, is decreasing, revealing higher viscous losses  $\nu_{\rm bundle}$ as the bundle forms, in accordance with (5).

In this experiment, the resonance frequency experiences a shift of 0.4 Hz that corresponds to an increase in the stiffness of  $12 \,\mathrm{mN} \cdot \mathrm{m}^{-1}$ , i.e., a bundle containing  $\sim 200 \,\lambda$ -DNA molecules

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[19]. (For the sake of clarity, it should be noted that the SNT used in this trapping experiment has different characteristics than those used in the feedback approach of Sections III and IV)

3) Sensor Resolution: Accurate biosensing by SNT requires high-resolution measurements. Thus, special care in signal conditioning (detection of small currents < 10 pA), noise reduction, and control of the experimental conditions (e.g., the stability of the meniscus air/biological liquid/tweezers) have allowed to sense resonance frequency shifts of 25 mHz and quality factor changes of 0.2. Based on this resolution, the minimum change that can be sensed due to the stiffness variation is in the range of 15 molecules of  $\lambda$ -DNA [20].

After trapping, the bundle of molecules is mechanically stimulated (in air or in a biological solution) with a sinusoidal signal at the resonance frequency of the system. Slight changes in the frequency response permit a fine characterization of the evolution of the mechanical properties of the molecule bundle revealing biological interactions.

#### III. DYNAMIC MODELING AND IMPROVEMENT OF THE PARAMETRIC SENSITIVITY BY A FEEDBACK APPROACH

In this Section, we describe how to reduce the closed-loop 216 resonance frequency of the tweezers in order to improve its 217 sensitivity to molecule stiffness variations. 218

#### A. Dynamic Modeling and Open-Loop Sensitivity 219 to Stiffness Variations 220

1) Modeling: Newton's second law applied to the damped 221 oscillator formed by the SNT+bundle of Fig. 2(d) can be for-222 mulated by 223

$$\dot{X} = \underbrace{\begin{bmatrix} 0 & 1 \\ -\frac{(k+k_{\text{bundle}}(t))}{M} & -\frac{(\nu+\nu_{\text{bundle}}(t))}{M} \end{bmatrix}}_{A} X$$

$$+\underbrace{\begin{bmatrix} 0 \\ \frac{1}{M} \end{bmatrix}}_{B} F_{\text{es}}$$
(6)

and 224

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$$y = \underbrace{\begin{bmatrix} 0 \ 1 \end{bmatrix}}_{C} X \tag{7}$$

where  $A \in \mathbb{R}^2, B \in \mathbb{R}^{(2,1)}, C \in \mathbb{R}^{(1,2)},$  X is the state vector (displacement & velocity,  $X = \begin{pmatrix} x \\ \dot{x} \end{pmatrix}$ ), and y is the output vector 226 or the measurement. 227

2) Identification: Prior to the development of an accurate control strategy, the model parameters [M, k, v], actuation, and sensor gains of (1)–(5)] need to be extracted from device measurements. These parameters are identified through standard recursive approach with least squares method from SNT responses to small signal actuation with a 20 V offset. Identified parameters are summarized in Table II.

TARIFII THEORETICAL AND IDENTIFIED MODEL PARAMETER VALUES OF THE SNT

Parameters	Theoretical	Identified
Mechanical parameters		
M (kg)	$360 \times 10^{-9}$	$360 \times 10^{-9}$
k (N·m <sup>-1</sup> )	$49.3^{a}$	24.9
$\nu \left( \text{Ns} \cdot \text{m}^{-1} \right)$	-	$100 \times 10^{-6}$
Comb-drive actuator		
$\alpha_{\rm es}  \left( {\rm N} \cdot {\rm V}^{-2} \right)$	$29.2 \times 10^{-9a}$	$35.5\times10^{-9}$
Capacitive sensor		
$\beta_{\rm C} \left( {\rm F \cdot m^{-1}} \right)$	$412.2 \times 10^{-9a}$	$227.5 \times 10^{-9}$

<sup>&</sup>lt;sup>a</sup> These values are calculated from the designed springs and electrodes widths and does not take into account the significant beam narrowing during etching

3) Open-Loop Sensitivity to Stiffness Variation: The minimum number of molecules that can be discerned by resonance frequency shifts due to the molecular stiffness variation is linked to the SNT sensitivity to stiffness variation  $\Delta k$ , i.e.,

$$\frac{\partial f_{\rm R}}{\partial k} = \frac{1}{4\pi\sqrt{kM}} \text{ or } \frac{1}{8\pi^2 M f_{\rm R}}.$$
 (8)

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This sensitivity is equal to  $26.4 \,\mathrm{Hz/(N\cdot m^{-1})}$  with the identified parameters of Table II. It appears that for a given molecular stiffness change,  $\Delta k$ , the effect on the resonance frequency variation  $\Delta f_{\rm R}$  is enhanced with low SNT stiffness, k, and, therefore, for low resonance frequency  $f_{\rm R}$ . In relation to Fig. 2, the current experimental resolution in the resonant frequency shift measurement is close to 25 mHz, a value that corresponds to  $1~\text{mN}\cdot\text{m}^{-1}$  or 15  $\lambda$ -DNA molecules. From (8), a reduction of  $f_{\rm R}$  by one order of magnitude; i.e. 2 orders of magnitude for k, would improve the detection threshold down to the single molecule level.

The development of a new MEMS device having a very low stiffness ( $< 0.2 \,\mathrm{N\cdot m^{-1}}$  remains extremely challenging in terms of fabrication and operation. Thus, we propose to reduce the overall stiffness by closed-loop control. In such a control strategy, the physical system (i.e., the SNT) remains the same but the closed-loop system SNT combined with the feedback law will have the desired designed dynamic behavior.

#### B. Improvement of the Parametric Sensitivity by a Feedback Approach

- 1) Control Strategy: The control design strategy is depicted 259 in the flowchart of Fig. 3. The aim of the feedback is to reduce the resonance frequency of the closed-loop system. To this end, we use a state feedback eigenstructure assignment in order to assign both the closed-loop resonance frequency and damping factor. This state feedback is then implemented using a Luenbergerkind observer. The originality of the observer we use is that it has to be designed in order to neither deteriorate the closed-loop sensitivity nor amplify the noise.
- 2) Eigenstructure Assignment Using State Feedback: The system is under its controllable canonical state-space representation (A, B, C) given by (6) and (7). The state feedback gain vector  $L = (l_1, l_2) \in \mathbb{R}^{(1,2)}$  is designed to assign the desired

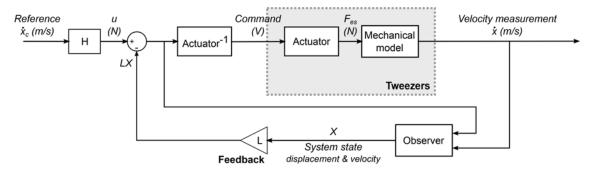


Fig. 3. Feedback control scheme. Gray box, called *Tweezers*, comprises the mechanical and the actuator models. Actuator square law needs to be inverted in order to linearize the control signal. The state feedback gain vector L and the feedforward gain H are designed to assign the desired closed-loop poles and to ensure unitary static gain. The observer is a Luenberger observer reconstructing the state of the system X for the state feedback. The sensor model is not shown here. The device output voltage is converted into the tip velocity by dividing by the sensor gain  $1/\beta_C$ .

TABLE III POLE PLACEMENT OF THE CLOSED-LOOP SYSTEM AND STATE FEEDBACK GAIN L According to the Resonance Frequency Reduction  $\boldsymbol{n}$ 

Freq.	$L~([\mathbf{N}\cdot\mathbf{m}^{-1}],[\mathbf{N}\mathbf{s}\cdot\mathbf{m}^{-1}])$	Poles
$f_{\rm R}$	(0,0)	$-138.9 \pm 8.3 \times 10^3 i$
$f_{\rm R} / 1.2$	$(-7.6, -16.7 \times 10^{-6})$	$-115.7 \pm 6.9 \times 10^3 i$
$f_{\rm R} / 2.0$	$(-18.7, -50.0 \times 10^{-6})$	$-69.4 \pm 4.2 \times 10^3 i$
$f_{\rm R} / 3.2$	$(-22.5, -68.8 \times 10^{-6})$	$-43.4 \pm 2.6 \times 10^3 i$
$f_{\rm R} / 5.0$	$(-23.9, -80.0 \times 10^{-6})$	$-27.8 \pm 1.7 \times 10^3 i$
$f_{\rm R}/10$	$(-24.7, -90.0 \times 10^{-6})$	$-13.9 \pm 0.8 \times 10^3 i$

closed-loop poles. The actuation of the system  $F_{\rm es}$  is equal to u-LX, and the closed-loop system model becomes

$$\dot{X} = \begin{bmatrix} 0 & 1\\ -\frac{k+l_1}{M} & -\frac{\nu+l_2}{M} \end{bmatrix} X + \begin{bmatrix} 0\\ \frac{1}{M} \end{bmatrix} u. \tag{9}$$

Thus, the closed-loop resonance frequency and sensitivity are given by

$$f_{\rm R-cl} = \frac{1}{2\pi} \sqrt{\frac{(k+l_1)}{M}}$$
 (10)

$$\frac{\partial f_{\text{R-cl}}}{\partial k} = \frac{1}{4\pi\sqrt{(k+l_1)M}} = \frac{1}{8\pi^2 M f_{\text{R-cl}}}.$$
 (11)

The state feedback gain L is, then, determined in order to assign the closed-loop poles in such a way that the system resonates at a lower frequency than the natural frequency of the tweezers (see Table III). L is computed to scale down this resonant frequency (initially  $1323.6~{\rm Hz}$ ), by a given factor depending on the desired sensitivity enhancement while keeping the damping factor unchanged.

When the closed-loop resonance frequency is scaled down by a factor n compared to the SNT natural resonance frequency, the sensitivity to small stiffness variation is improved with the same ratio according to (11). Fig. 4 shows the computed frequency shift of the closed-loop system caused by a significant stiffness variation for different frequency scaling down factor, n.

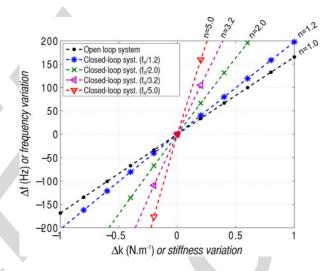


Fig. 4. Evolution of the resonance frequency shift  $\Delta f_R$  and  $\Delta f_{R-cl}$  with the added stiffness  $\Delta k$  variation for different frequency reduction ratio n.

#### C. Observer Design

The implementation of the aforementioned state feedback is done through the use of a Luenberger observer. The observer is designed such that it preserves the good closed-loop sensitivity to parametric variations and it does not amplify too much noise.

The poles of the observer have then to be carefully chosen. We used the following guidelines.

- (1) The observer is designed to be faster than the original device—with poles at least two times faster than the poles of the system in open loop (cf., Table III).
- (2) The observer poles are chosen not too fast in order to avoid excessive noise amplification.
- (3) The final location of the observer poles is chosen in order to preserve the sensitivity of the system to stiffness variations (by using root locus).

In the following section, simulations illustrate 1) the relevance of the method and 2) the importance of the design of the observer towards the enhancement of the sensitivity.

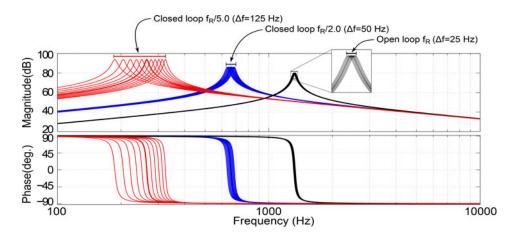


Fig. 5. Simulated frequency results for open-loop driven system and two different closed-loop systems. The three systems undergo a variation of k the system stiffness. k vary from -0.5 to 0.5 N/m by 0.1 N/m step. In black line, the tweezer transfer function, i.e., the open-loop system. In blue line, the implemented (n=2) closed-loop system transfer function. The system resonates at a frequency twice lower. In red line, the implemented (n=5) closed-loop system transfer function. The system resonates at a frequency five times lower.

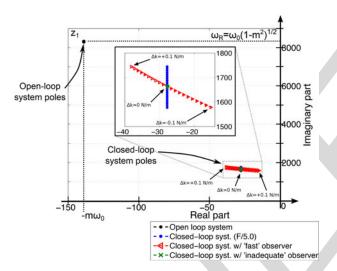


Fig. 6. Root locus of the closed-loop systems with n=5 and with different observers. Complex conjugate poles are plotted in the positive imaginary half s plan. The parameter k varies from -0.1 to 0.1 N/m by 0.01 N/m steps. The black dots show the poles of the open-loop driven system. The ideal pole path of the closed-loop system due to the k variations are plotted with blue dots (i.e., no observer). For different observers designs, poles dependencies (and equivalent resonance frequency  $\omega_{\rm R}=2\pi f_{\rm R}$  and damping m) on the tweezers stiffness k are shown by red triangle and green cross plots. NB: Green cross dots path are imperceptible since they are at this scale all concentrate in the middle.

#### 308 D. Simulation Results

From parameters identified in Table II, the feedback scheme (see Fig. 3) has been first programmed and tested under MAT-LAB/Simulink without implementing the observer (i.e., all the state variables are supposed to be measured). Fig. 5 compares the resonance frequency shift between the open-loop driven system and the closed-loop systems computed for two resonance frequency reduction factors set to n=2 and 5. As expected, the sensitivity of the closed-loop system is improved proportionally to n.

In a second instance, we implemented two different kinds of observers (with different dynamics); Fig. 6 illustrates two cases.

TABLE IV
POLE PLACEMENT OF THE OBSERVER FOR STATE FEEDBACK IMPLEMENTATION

Open-loop system (Tweezers)	$z_{1,2} = -138.9 \pm 8.3 \times 10^3  i$
closed-loop system (n = 5) Observer 1 (fast) Observer 2 (slow) Observer 3 (inadequate)	$z_{1,2} = -27.8 \pm 1.7 \times 10^{3} i$ $z_{1,2} = -277.8 \pm 0.2 \times 10^{3} i$ $z_{1,2} = -27.8 \pm 0.2 \times 10^{3} i$ $z_{1,2} = -277.8 \pm 16.7 \times 10^{3} i$

This Figure shows that the root locus (due to *k* parameter variations) is deformed by the observer dynamics. The sensitivity of the resonance frequency to stiffness variations is given by the amplitude of the imaginary axis variations. Table IV reports the poles of the observers for which simulation results are presented.

In the insert of Fig. 6, the poles of the system implemented with the observer named *inadequate* do not move compared to the ideal root locus (ideal state feedback case). If the poles of the observer are inappropriately chosen, the sensitivity of the poles to variations of k is significantly reduced. Observer poles close to the real axis, i.e., with an imaginary part close to 0, are the more appropriate to reconstruct the displacement state and to preserve the sensitivity of the system to variations of k. The socalled *fast* observer allows us to increase the resonance frequency shift ( $\omega_R = 2\pi f_R$  in Fig. 6) close to the performances theoretically expected.

### IV. EXPERIMENTAL APPLICATION FOR EXTRA-STIFFNESS CHARACTERIZATION

The previous method has been experimentally tested with the mechanical characterization of biomolecules of *fibronectin*. The physical sensor is the SNT, while the feedback controller and the observer are implemented in a dSPACE prototyping board.

#### A. Experimental Protocol

Fibronectin is a protein, 100 nm in length, of the extracellular matrix which plays a major role in cell adhesion, growth,

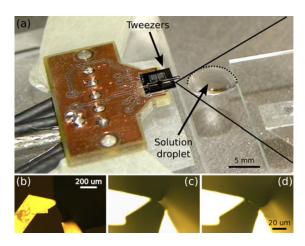


Fig. 7. Experimental setup for the trapping of *fibronectin* molecules. (a) The SNT are fixed. A droplet of *fibronectin* solution is dropped on a glass slice which is mounted on the XYZ stage. (b)–(d) Sequences of the molecule combing. Under optical microscopy, the tweezers' tips are retrieved from the solution meniscus. A bundle of *fibronectin* is released between the tips.

migration, and differentiation [33]. These molecules are used to polymerize the fibrillar matrix at the surface of the cell. We have used this property to trap by combing a bundle of *fibronectin*. After immersion of the tips in the solution, they are retrieved by the side forming a polymerized bundle of molecules (see Fig. 7).

The closed-loop systems are evaluated with *fibronectin* bundle in air, in dry condition. This stable condition was preferred to the immersed one as we can avoid any effect of liquid surface tension stability that can interfere with the sole system response. For each case, a sinusoidal reference of  $0.02\,V_{\rm AC}$  with an offset of  $20\,V_{\rm DC}$  is applied to the system and the oscillation frequencies are spanned from 1000 to  $2000~{\rm Hz}$  with a step of  $2~{\rm Hz}.$  Then, the measurement is performed with the lock-in amplifier.

Two sets of experiments have been performed with two different bundles. From the resonance frequency shifts in open loop, the stiffness of the first bundle is  $11.8~\rm N\cdot m^{-1}$ , and  $4.3~\rm N\cdot m^{-1}$  for the second one. In total, 22 resonance responses are recorded and compared.

#### B. Results and Discussions

1) Experimental Results: Fig. 8 shows the frequency responses of four different systems with and without a bundle of fibronectin.

In Fig. 8(a), the responses of the open-loop driven SNT are given. The dashed-line curve is the reference curve, which corresponds to the frequency response of the SNT alone driven in open loop. The solid line is the response of the tweezers with the bundle trapped in between the tips. The resonance frequency increases and the amplitude decreases due to the additional stiffness and loss contributions of the molecules. In Fig. 8(b)–(d), the results of the closed-loop systems with desired reduction ratios are of 1.1, 1.2 and 1.5.

Resonance responses are fitted with a second-order resonator model and characteristics such as the resonance frequency are extracted. The Table V sums up the resonance frequencies for the experiment with/without the first bundle for five different

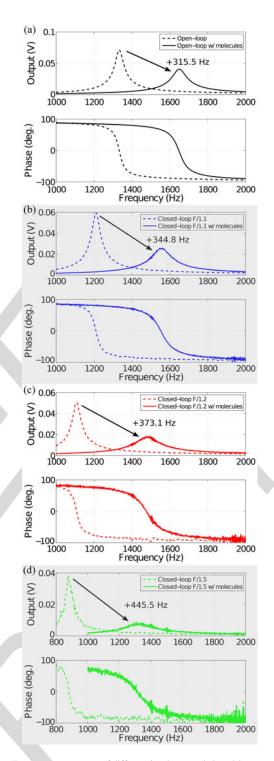


Fig. 8. Frequency responses of different implemented closed-loop systems for 0.02 V sinusoidal signal with an offset of 20 V. In dotted lines, the responses of the systems with the bundle of molecules. In plain lines, the responses of the systems without molecules. (a) Open-loop driven systems. (b) Closed-loop systems implemented with  $f_{\rm R}/1.1$ . (c) Closed-loop systems implemented with  $f_{\rm R}/1.2$ . (d) Closed-loop systems with  $f_{\rm R}/1.5$ .

systems. Through reducing the resonance frequency of the system, the measured results prove the increase of the frequency shift due to the extra-stiffness of the bundle.

Moreover Fig. 9 tends to 1) confirm with second set of experiments, the enhancement of the sensitivity through the proposed

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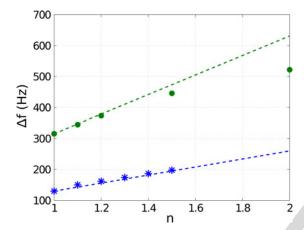
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TABLE V EXPERIMENTAL RESULTS FOR THE BUNDLE 1 (RELATED TO FIG. 8)

System (n)	Freq. w/o bundle (Hz)	Freq. w/ bundle (Hz)	Freq. shift (Hz)
Open loop (1.0)	1333.6	1649.1	+315.5
Closed-loop (1.1)	1211.1	1555.9	+344.8
Closed-loop (1.2)	1109.1	1482.2	+373.1
Closed-loop (1.5)	881.9	1327.4	+445.5
Closed-loop (2.0)	653.9	1175.6	+521.7



Synthesis of the experimental results. In abscissa, is informed the reduction factor (n) applied to the resonance frequency for the closed-loop system. The shift  $\Delta f$  is the resonance frequency difference due to the presence of the bundle of molecules in between the tips. The dotted lines represent the theoretical improvement expected. The two sets of results obtained with two different bundles are plotted. The green dots correspond to the experiment of Fig 8 and TableV (bundle of  $11.8 \text{ N} \cdot \text{m}^{-1}$  stiffness). The blue stars corresponds to the second bundle of  $4.3 \text{ N} \cdot \text{m}^{-1}$  stiffness.

feedback strategy and 2) show a good agreement with the theory. Small differences are discussed in the following section.

2) Discussion: The synthesis of Fig. 9 demonstrates the enhancement of the frequency shifts in agreement with the theory. However, the enhancement for closed-loop system with a reduction factor of 2.0 is lower than expected, and the implementation for a higher reduction factor has not been achieved. In this case, the system tends to become unstable. We mainly associated this phenomenon to the observer implementation and to the presence of some delays in the feedback loop.

We have demonstrated previously that observer's dynamic changes the root locus of system's poles, the behavior of the closed-loop system under parameter variations and impacts the performances of the method. Furthermore, delays have been characterized due to D/A and A/D converters dead times and to the phase shift brought by the current amplifiers. Current works are devoted to the precise characterization of such delays and to their integration in the control design process.

#### V. CONCLUSION AND FUTURE WORK

The present study investigates the relevance of a feedback approach in the improvement of the performances of MEMS tweezers for the detection and the mechanical characterization of biological molecules. An improvement of 50% on the sensitivity

to mechanical stiffness variations has been reached theoretically and experimentally on *fibronectin* bundles through the emulation of a more compliant system. The stiffness have been brought 410 down to 10.5 N/m instead of originally 24.5 N/m.

The main drawback of this approach is the sensitivity of 412 the observer to the stiffness variations. Unusually, the poles of the observer are uncontrollable resulting from the separation 414 principle. During the sensing of the molecules stiffness, the 415 separation principle is not satisfied anymore and the observer 416 poles can become unstable. We are actually working on selfscheduled observers aiming at adapting the observer dynamics 418 with respect to sensed parametric variations. It should solve the main drawback of this promising approach and allows us to go one step further into the theoretical closed-loop performances.

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# Improvement of Silicon Nanotweezers Sensitivity for Mechanical Characterization of Biomolecules Using Closed-Loop Control

Nicolas Lafitte, Yassine Haddab, Yann Le Gorrec, *Member, IEEE*, Hervé Guillou, Momoko Kumemura, Laurent Jalabert, Dominique Collard, *Member, IEEE*, and Hiroyuki Fujita, *Member, IEEE* 

Abstract—In this paper, we show that closed-loop control can be advantageously used for the characterization of mechanical properties of biomolecules using silicon nanotweezers (SNT). SNT have already been used in open-loop mode for the characterization of mechanical properties of DNA molecules. Up to now, such an approach allows the detection of stiffness variations equivalent to about 15 DNA molecules. Here, it is shown that this resolution is inversely proportional to the resonance frequency of the whole system and that real-time feedback control with state observer can drastically improve the performances of the tweezers used as biosensors. Such improvement is experimentally validated in the case of the manipulation of fibronectin molecules. The results are promising for the accurate characterization of biopolymers such as DNA molecules.

*Index Terms*—Biosensor, MEMS tweezers, parameter detection, sensitivity improvement, state feedback.

#### I. INTRODUCTION

RECENT developments in micronano manipulation tools have revealed crucial information on the mechanical behavior of biomolecules [1]–[3]. These manipulations generally performed on a single molecule have given the quantitative data needed to elucidate fundamental biological processes as DNA wrapping [4] and replication [5], or cell cytoskeleton dynamics through actin filament [6], and microtubule mechanical responses [7].

Several methods are already available for performing biological experiments at the molecular level: magnetic tweezers [8], [9], optical tweezers [10], AFM cantilevers [11], and microfibers [2], [12]. A single filamentary macromolecule, such as DNA, is first attached to a surface at one end, and to a bead

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or a flexible cantilever at the other end. Forces or displacements are sensed by optical measurements. Nevertheless, the real-time operation and the routine implementation of these techniques remain difficult to achieve, as they require complex experimental procedures. In this respect, microelectromechanical systems (MEMS) offer an advantage for systematic analysis since accurate molecular level tools (actuator, end effectors, and sensor) can be integrated on a MEMS platform. Furthermore, these devices can be produced at low cost with batch fabrication, as have the microgrippers developed to grab microsized objects [13] or cells [14]. However, up to now, the performances of such tools are not sufficient to deal with single molecule characterization. In previous reports, silicon nanotweezers (SNT) were designed and fabricated to trap DNA molecules [15], [16] and characterize DNA bundles [17] and reactions on DNA [18]. The stiffness of DNA bundles was derived from the open-loop measurement of the mechanical resonance frequency shift. The current sensing resolution is in the range of  $1 \text{ mN} \cdot \text{m}^{-1}$ , which corresponds to approximatively 15  $\lambda$ -DNA molecules [20]. This resolution is limited by the design of the tweezers and the noise of the electronic instrumentation. To improve the overall system sensitivity towards lower stiffness, the primary route consists of reducing the tweezers stiffness to bring the sensor and measured object characteristics close to the same range. Unfortunately, such a design would result in a system that would be too fragile to be processed and utilized.

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Our approach consists of scaling down the resonance frequency of the closed-loop system to enhance the sensitivity to SNT stiffness change. After a proper identification of the SNT model, the feedback control has been simulated, optimized, and then implemented. This control strategy has been evaluated with thin *fibronectin* molecular bundles trapped between SNT tips, and we propose to discuss the advantages and the limits of such closed-loop operations.

This paper is organized as follows. Section II describes the tweezers, the sensing principle, and provides an example of monitoring the DNA trapping. The dynamic model of the tweezers is then detailed in Section III-A. The closed-loop control strategy and its potential to improve the system sensitivity are discussed in Sections III-B and III-C. Section IV details the experimental implementation and provides information on *fibronectin* molecules measurement with different

 $^1\lambda\text{-DNA}$  is DNA of the bacteriophage  $\lambda$  having a contour length of  $16.4~\mu m$  and a stiffness in elastic regime of  $66~\mu N \cdot m^{-1}$  in low ionic solution [19].

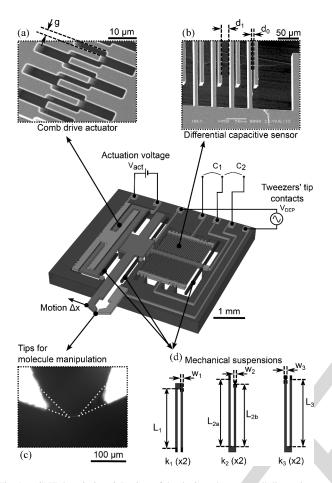


Fig. 1. SNT description. 3-D view of the device whose overall dimensions are  $4.5 \times 4.5 \times 0.5 \,\mathrm{mm}$ . The mobile parts are shown in red and the mechanical suspensions in blue. (a) Scanning electronic microscopic (SEM) image of the actuator consisting of 880 pairs of interdigitated combs separated by a 2- $\mu\mathrm{m}$  gap, and organized in two series. (b) SEM image of the integrated sensor of two opposing series of 30 combs. (c) Microscope image of the tweezers tips with a DNA bundle in between. (d) Design of the three sets of suspensions,  $k_1$  for the actuator polarization,  $k_2$  for the mobile arm, and  $k_3$  for central sensor plate.

closed-loop settings. Closing remarks and perspectives are given in Section V.

## II. DESCRIPTION OF THE DEVICE AND ITS USE FOR THE MECHANICAL CHARACTERIZATION OF BIOMOLECULES BUNDLES

#### A. Device Description

1) Design: Fig. 1 shows a three-dimensional illustration of the SNT device. It consists of two arms terminated in sharp tips that act as electrodes to trap biomolecules by dielectrophoresis (DEP) forces [15]. One tip is fixed; the other one is moved by an electrostatic actuator. The motion of the mobile electrode is measured by capacitances whose value linearly varies with electrode displacement; two variable capacitances are mounted in differential mode [21].

2) Electrostatic Actuation: The mobile arm is actuated by attractive electrostatic forces generated in an interdigitated comb drive [see Fig. 1(a)], one of the most widely used architecture

in MEMS design [22], [23]. The maximal stroke is limited compared to that achievable by a parallel plate actuator but the generated force only depends on the actuation voltage and not on the electrode position, thus, enabling a simpler actuation control. The force is given by (1) where  $\varepsilon_0$  is the vacuum permittivity (we consider the relative permittivity of air = 1),  $N_a$  is the number of comb fingers, t is the device thickness, t is the lateral gap between fingers, and t is the actuation voltage. Due to its capacitive structure, the comb drive generates attractive forces between its two electrodes

$$F_{\rm es} = \frac{1}{2} \frac{\varepsilon_0 \times N_a \times t}{g} V_{\rm act}^2 = \alpha_{\rm es} \times V_{\rm act}^2.$$
 (1)

3) Mechanical Structure: The mobile part of the tweezers is linked to the silicon frame by flexible beams [see Fig. 1(d)]. Commonly integrated in mechanical microsystems [23], [24], folded beam springs are designed to minimize beam areas, decrease their mechanical stiffness and enhance displacement ranges [25]. In the SNT design of Fig. 1, three sets of folded beam suspensions support the mobile part of the system (the comb-drive actuator, the mobile tip, and the capacitive sensor) and provide the electrical connections for the actuation and the sensing. The springs are arranged symmetrically along the actuation and sensing axis to minimize any rotation, and their sum gives the total stiffness k of the device.

A highly compliant system is required to sense the mechanical characteristic of trapped molecules on the tweezers response. On the other hand, a minimum stiffness is mandatory, 1) to endure the fabrication processes and manipulations, 2) to support the mobile system weight, and 3) to prevent sticking due to attractive surface forces between the comb-drive actuator and the capacitive sensor electrodes.

4) Displacement/Velocity Sensing: The tip position is measured by a capacitive sensor designed in a triplate configuration with transverse combs [see Fig. 1(b)] [26]. The central electrode, linked to the mobile arm, moves in between two fixed electrodes creating the two differential capacitances  $C_1$  and  $C_2$  whose difference,  $\Delta C$ , is related to the displacement x. For small displacements (i.e., x is much smaller than the gaps between electrodes  $d_0$  and  $d_1$ ),  $\Delta C$  is proportional to x [17],  $N_b$  is the number of capacitance electrodes, and L the length of the electrodes

$$\Delta C = C_1 - C_2 \tag{2}$$

$$\simeq 2\varepsilon_0 N_b Lt \left(\frac{1}{d_0^2} - \frac{1}{d_1^2}\right) x = \beta_{\rm C} \times x$$
 (3)

5) Electronic Read-Out: The tweezers motion is sensed by the measurement of the differential capacitance  $\Delta C$  (2) through current sensing. In dynamic mode, a dc voltage ( $V_{\rm polar}$ ) is applied on the mobile central electrode, whose motion generates dynamic currents  $i_1$  and  $i_2$  flowing through the capacitances  $C_1$  and  $C_2$ , respectively, [27], [28]. The resulting currents related to the motion velocity are converted into voltages  $V_1$  and  $V_2$  by two low-noise current-to-voltage (A/V) preamplifiers (Signal Recovery, model 5182). The low input impedance of the preamplifier (virtual ground) ensures an accurate current conversion [17], [29].

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TABLE I NUMERICAL VALUES OF DIMENSIONS AND PARAMETERS OF THE SNT

Silicon thickness	
t (μm)	30
Comb drive actuator	
$N_a$	440
$g (\mu m)$	2
Mechanical suspensions	
$L_1, L_{2a} (\mu m)$	900
$L_{2b}, L_3 (\mu m)$	1000
$w_1, w_2, w_3 \; (\mu \text{m})$	15
Capacitive sensor	
$N_b$	30
$L (\mu m)$	585
$d_0(\mu m)$	5
$d_1 (\mu \text{m})$	20

The device dimensions are defined in Fig. 1.

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Finally, in harmonic mode, when the tweezers is moved by a sinusoidal actuation, a lock-in amplifier (NF, model LI 5640) performs the low-noise detection of the differential signal (amplitude and phase of  $V_1 - V_2$ ) at the actuation frequency. The reference signal is the motion frequency imposed by the actuation voltage.

All dimensions and parameters that has been considered for a proper model of the SNT are summed up in the TableI. Furthermore, Young's modulus and the density of the silicon are, respectively, 165 GPa and 2329 kg·m<sup>-3</sup>, and the permittivity  $\varepsilon_0 \text{ is } 8.85 \times 10^{-12} \text{ kg}^{-1} \cdot \text{m}^{-3} \cdot \text{A}^2 \cdot \text{s}^4.$ 

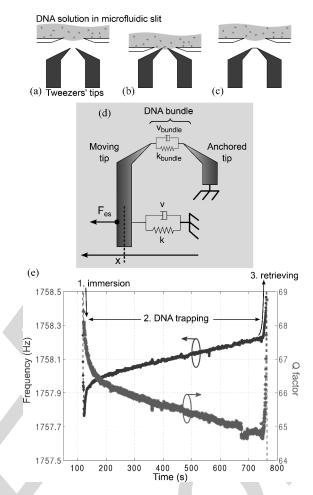
#### B. Monitoring Biomolecules Manipulation With SNT

This Section presents the way that the SNT are used as biosensors and points out the measurement sensitivity requirements. This example is illustrated with DNA but the same technique has been applied to gelatin [30] and microtubules [31].

1) DNA Bundle Trapping by DEP: The trapping is achieved by applying an ac electric field  $(E = 1 \text{ MV} \cdot \text{m}^{-1})$  between the two opposing tips of the tweezers [32] once partially immersed in the solution. The retrieving of the DNA bundle is routinely performed from a droplet solution [15], [17], and single molecule trapping has even been demonstrated with a pulsed DEP signal [16].

In [20], an improved method is proposed in which the tips are introduced into an open fluidic cavity optimized to guaranty a stable meniscus. The SNT position is fixed and the cavity is mounted on a XYZ stage controlled by programmable robot for repeatable and controlled tip immersion, as illustrated in Fig. 2(a)–(c).

2) Real-Time Monitoring: The SNT characterization allows monitoring of the DNA trapping in real time. The characterization of the SNT in real time allows the monitoring of the DNA trapping. The number of trapped DNA molecules during the DEP is controlled by the evolution of the resonance frequency and quality factor of the system (Tweezers+DNA bundle). The SNT are modeled by a linear second-order model as depicted in Fig. 2(d). The resonance frequency,  $f_R$ , and the quality factor



λ-DNA molecules trapping with SNT. The first three sketches illus-Fig. 2. trate the key phases of the experiment: (a) the immersion of the tips in the DNA solution, (b) the DNA trapping by DEP, and (c) the retrieving of DNA bundle between the tips. (d) Equivalent dynamic model of the SNT with bundle. (e) Evolution of the SNT+bundle resonance frequency and Q factor during the trapping.

Q of the SNT+bundle damped oscillator are defined by

$$f_{\rm R}(t) = \frac{1}{2\pi} \sqrt{\frac{(k + k_{\rm bundle}(t))}{M}}$$

$$Q(t) = \frac{\sqrt{(k + k_{\rm bundle}(t)) \times M}}{(\nu + \nu_{\rm bundle}(t))}$$
(5)

$$Q(t) = \frac{\sqrt{(k + k_{\text{bundle}}(t)) \times M}}{(\nu + \nu_{\text{bundle}}(t))}$$
 (5)

where M is the mass of the movable tip, k is the stiffness of the suspensions, and  $\nu$  is the equivalent viscous losses.  $k_{\rm bundle}$  and  $\nu_{\rm bundle}$  are the time varying bundle stiffness and viscosity; the mass of the bundle is neglected compared to M.

As shown in Fig. 2(e), as the DEP trapping progresses, the system moves to higher resonance frequencies indicating an increase in the bundle stiffness according to (4). At the same time, the signal amplitude at resonance, proportional to the quality factor Q, is decreasing, revealing higher viscous losses  $\nu_{\rm bundle}$ as the bundle forms, in accordance with (5).

In this experiment, the resonance frequency experiences a shift of 0.4 Hz that corresponds to an increase in the stiffness of  $12 \,\mathrm{mN} \cdot \mathrm{m}^{-1}$ , i.e., a bundle containing  $\sim 200 \,\lambda$ -DNA molecules

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[19]. (For the sake of clarity, it should be noted that the SNT used in this trapping experiment has different characteristics than those used in the feedback approach of Sections III and IV)

3) Sensor Resolution: Accurate biosensing by SNT requires high-resolution measurements. Thus, special care in signal conditioning (detection of small currents < 10 pA), noise reduction, and control of the experimental conditions (e.g., the stability of the meniscus air/biological liquid/tweezers) have allowed to sense resonance frequency shifts of 25 mHz and quality factor changes of 0.2. Based on this resolution, the minimum change that can be sensed due to the stiffness variation is in the range of 15 molecules of  $\lambda$ -DNA [20].

After trapping, the bundle of molecules is mechanically stimulated (in air or in a biological solution) with a sinusoidal signal at the resonance frequency of the system. Slight changes in the frequency response permit a fine characterization of the evolution of the mechanical properties of the molecule bundle revealing biological interactions.

#### III. DYNAMIC MODELING AND IMPROVEMENT OF THE PARAMETRIC SENSITIVITY BY A FEEDBACK APPROACH

In this Section, we describe how to reduce the closed-loop 216 resonance frequency of the tweezers in order to improve its 217 sensitivity to molecule stiffness variations. 218

#### A. Dynamic Modeling and Open-Loop Sensitivity 219 to Stiffness Variations 220

1) Modeling: Newton's second law applied to the damped 221 oscillator formed by the SNT+bundle of Fig. 2(d) can be for-222 mulated by 223

$$\dot{X} = \underbrace{\begin{bmatrix} 0 & 1 \\ -\frac{(k+k_{\text{bundle}}(t))}{M} - \frac{(\nu+\nu_{\text{bundle}}(t))}{M} \end{bmatrix}}_{A} X$$

$$+ \underbrace{\begin{bmatrix} 0 \\ \frac{1}{M} \end{bmatrix}}_{B} F_{\text{es}}$$
(6)

and 224

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$$y = \underbrace{\begin{bmatrix} 0 \ 1 \end{bmatrix}}_{C} X \tag{7}$$

where  $A \in \mathbb{R}^2, B \in \mathbb{R}^{(2,1)}, C \in \mathbb{R}^{(1,2)},$  X is the state vector (displacement & velocity,  $X = \begin{pmatrix} x \\ \dot{x} \end{pmatrix}$ ), and y is the output vector 226 or the measurement. 227

2) Identification: Prior to the development of an accurate control strategy, the model parameters [M, k, v], actuation, and sensor gains of (1)–(5)] need to be extracted from device measurements. These parameters are identified through standard recursive approach with least squares method from SNT responses to small signal actuation with a 20 V offset. Identified parameters are summarized in Table II.

TARIFII THEORETICAL AND IDENTIFIED MODEL PARAMETER VALUES OF THE SNT

Parameters	Theoretical	Identified
Mechanical parameters		
M (kg)	$360 \times 10^{-9}$	$360 \times 10^{-9}$
k (N·m <sup>-1</sup> )	$49.3^{a}$	24.9
$\nu \left( \text{Ns} \cdot \text{m}^{-1} \right)$	-	$100 \times 10^{-6}$
Comb-drive actuator		
$\alpha_{\rm es}  \left( { m N} \cdot { m V}^{-2} \right)$	$29.2 \times 10^{-9a}$	$35.5 \times 10^{-9}$
Capacitive sensor		
$\beta_{\rm C} \left( {\rm F \cdot m^{-1}} \right)$	$412.2 \times 10^{-9a}$	$227.5 \times 10^{-9}$

<sup>&</sup>lt;sup>a</sup> These values are calculated from the designed springs and electrodes widths and does not take into account the significant beam narrowing during etching

3) Open-Loop Sensitivity to Stiffness Variation: The minimum number of molecules that can be discerned by resonance frequency shifts due to the molecular stiffness variation is linked to the SNT sensitivity to stiffness variation  $\Delta k$ , i.e.,

$$\frac{\partial f_{\rm R}}{\partial k} = \frac{1}{4\pi\sqrt{kM}} \text{ or } \frac{1}{8\pi^2 M f_{\rm R}}.$$
 (8)

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This sensitivity is equal to  $26.4 \,\mathrm{Hz/(N\cdot m^{-1})}$  with the identified parameters of Table II. It appears that for a given molecular stiffness change,  $\Delta k$ , the effect on the resonance frequency variation  $\Delta f_{\rm R}$  is enhanced with low SNT stiffness, k, and, therefore, for low resonance frequency  $f_{\rm R}$ . In relation to Fig. 2, the current experimental resolution in the resonant frequency shift measurement is close to 25 mHz, a value that corresponds to  $1 \text{ mN} \cdot \text{m}^{-1}$  or 15  $\lambda$ -DNA molecules. From (8), a reduction of  $f_{\rm R}$  by one order of magnitude; i.e. 2 orders of magnitude for k, would improve the detection threshold down to the single molecule level.

The development of a new MEMS device having a very low stiffness ( $< 0.2 \,\mathrm{N\cdot m^{-1}}$  remains extremely challenging in terms of fabrication and operation. Thus, we propose to reduce the overall stiffness by closed-loop control. In such a control strategy, the physical system (i.e., the SNT) remains the same but the closed-loop system SNT combined with the feedback law will have the desired designed dynamic behavior.

#### B. Improvement of the Parametric Sensitivity by a Feedback Approach

- 1) Control Strategy: The control design strategy is depicted 259 in the flowchart of Fig. 3. The aim of the feedback is to reduce the resonance frequency of the closed-loop system. To this end, we use a state feedback eigenstructure assignment in order to assign both the closed-loop resonance frequency and damping factor. This state feedback is then implemented using a Luenbergerkind observer. The originality of the observer we use is that it has to be designed in order to neither deteriorate the closed-loop sensitivity nor amplify the noise.
- 2) Eigenstructure Assignment Using State Feedback: The system is under its controllable canonical state-space representation (A, B, C) given by (6) and (7). The state feedback gain vector  $L = (l_1, l_2) \in \mathbb{R}^{(1,2)}$  is designed to assign the desired

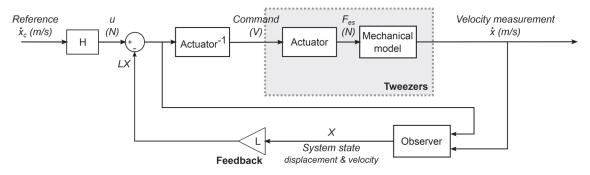


Fig. 3. Feedback control scheme. Gray box, called *Tweezers*, comprises the mechanical and the actuator models. Actuator square law needs to be inverted in order to linearize the control signal. The state feedback gain vector L and the feedforward gain H are designed to assign the desired closed-loop poles and to ensure unitary static gain. The observer is a Luenberger observer reconstructing the state of the system X for the state feedback. The sensor model is not shown here. The device output voltage is converted into the tip velocity by dividing by the sensor gain  $1/\beta_C$ .

TABLE III POLE PLACEMENT OF THE CLOSED-LOOP SYSTEM AND STATE FEEDBACK GAIN L According to the Resonance Frequency Reduction n

Freq.	$L~([\mathrm{N}\cdot\mathrm{m}^{-1}],[\mathrm{N}\mathrm{s}\cdot\mathrm{m}^{-1}])$	Poles
$f_{\rm R}$	(0,0)	$-138.9 \pm 8.3 \times 10^3 i$
$f_{\rm R} / 1.2$	$(-7.6, -16.7 \times 10^{-6})$	$-115.7 \pm 6.9 \times 10^3 i$
$f_{\rm R} / 2.0$	$(-18.7, -50.0 \times 10^{-6})$	$-69.4 \pm 4.2 \times 10^3 i$
$f_{\rm R} / 3.2$	$(-22.5, -68.8 \times 10^{-6})$	$-43.4 \pm 2.6 \times 10^3 i$
$f_{\rm R} / 5.0$	$(-23.9, -80.0 \times 10^{-6})$	$-27.8 \pm 1.7 \times 10^3 i$
$f_{\rm R} / 10$	$(-24.7, -90.0 \times 10^{-6})$	$-13.9 \pm 0.8 \times 10^3 i$

closed-loop poles. The actuation of the system  $F_{\rm es}$  is equal to u-LX, and the closed-loop system model becomes

$$\dot{X} = \begin{bmatrix} 0 & 1\\ -\frac{k+l_1}{M} & -\frac{\nu+l_2}{M} \end{bmatrix} X + \begin{bmatrix} 0\\ \frac{1}{M} \end{bmatrix} u. \tag{9}$$

Thus, the closed-loop resonance frequency and sensitivity are given by

$$f_{\rm R-cl} = \frac{1}{2\pi} \sqrt{\frac{(k+l_1)}{M}}$$
 (10)

$$\frac{\partial f_{\text{R-cl}}}{\partial k} = \frac{1}{4\pi\sqrt{(k+l_1)M}} = \frac{1}{8\pi^2 M f_{\text{R-cl}}}.$$
 (11)

The state feedback gain L is, then, determined in order to assign the closed-loop poles in such a way that the system resonates at a lower frequency than the natural frequency of the tweezers (see Table III). L is computed to scale down this resonant frequency (initially  $1323.6~{\rm Hz}$ ), by a given factor depending on the desired sensitivity enhancement while keeping the damping factor unchanged.

When the closed-loop resonance frequency is scaled down by a factor n compared to the SNT natural resonance frequency, the sensitivity to small stiffness variation is improved with the same ratio according to (11). Fig. 4 shows the computed frequency shift of the closed-loop system caused by a significant stiffness variation for different frequency scaling down factor, n.

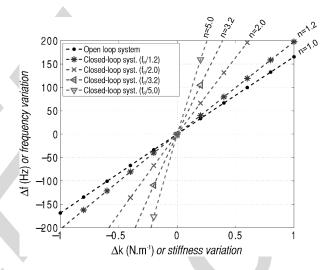


Fig. 4. Evolution of the resonance frequency shift  $\Delta f_R$  and  $\Delta f_{R-cl}$  with the added stiffness  $\Delta k$  variation for different frequency reduction ratio n.

#### C. Observer Design

The implementation of the aforementioned state feedback is done through the use of a Luenberger observer. The observer is designed such that it preserves the good closed-loop sensitivity to parametric variations and it does not amplify too much noise.

The poles of the observer have then to be carefully chosen. We used the following guidelines.

- (1) The observer is designed to be faster than the original device—with poles at least two times faster than the poles of the system in open loop (cf., Table III).
- (2) The observer poles are chosen not too fast in order to avoid excessive noise amplification.
- (3) The final location of the observer poles is chosen in order to preserve the sensitivity of the system to stiffness variations (by using root locus).

In the following section, simulations illustrate 1) the relevance of the method and 2) the importance of the design of the observer towards the enhancement of the sensitivity.

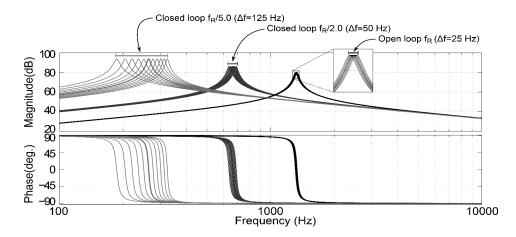


Fig. 5. Simulated frequency results for open-loop driven system and two different closed-loop systems. The three systems undergo a variation of k the system stiffness. k vary from -0.5 to 0.5 N/m by 0.1 N/m step. In black line, the tweezer transfer function, i.e., the open-loop system. In blue line, the implemented (n=2) closed-loop system transfer function. The system resonates at a frequency twice lower. In red line, the implemented (n=5) closed-loop system transfer function. The system resonates at a frequency five times lower.

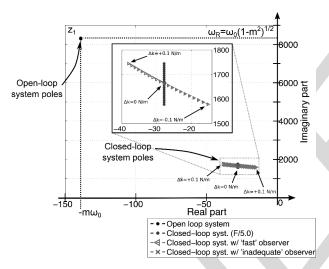


Fig. 6. Root locus of the closed-loop systems with n=5 and with different observers. Complex conjugate poles are plotted in the positive imaginary half s plan. The parameter k varies from -0.1 to 0.1 N/m by 0.01 N/m steps. The black dots show the poles of the open-loop driven system. The ideal pole path of the closed-loop system due to the k variations are plotted with blue dots (i.e., no observer). For different observers designs, poles dependencies (and equivalent resonance frequency  $\omega_{\rm R}=2\pi f_{\rm R}$  and damping m) on the tweezers stiffness k are shown by red triangle and green cross plots. NB: Green cross dots path are imperceptible since they are at this scale all concentrate in the middle.

#### 308 D. Simulation Results

From parameters identified in Table II, the feedback scheme (see Fig. 3) has been first programmed and tested under MAT-LAB/Simulink without implementing the observer (i.e., all the state variables are supposed to be measured). Fig. 5 compares the resonance frequency shift between the open-loop driven system and the closed-loop systems computed for two resonance frequency reduction factors set to n=2 and 5. As expected, the sensitivity of the closed-loop system is improved proportionally to n.

In a second instance, we implemented two different kinds of observers (with different dynamics); Fig. 6 illustrates two cases.

TABLE IV
POLE PLACEMENT OF THE OBSERVER FOR STATE FEEDBACK IMPLEMENTATION

Open-loop system (Tweezers)	$z_{1,2} = -138.9 \pm 8.3 \times 10^3  i$
Observer 1 (fast) Observer 2 (slow) Observer 3 (inadequate)	$z_{1,2} = -27.8 \pm 1.7 \times 10^{3} i$ $z_{1,2} = -277.8 \pm 0.2 \times 10^{3} i$ $z_{1,2} = -27.8 \pm 0.2 \times 10^{3} i$ $z_{1,2} = -277.8 \pm 16.7 \times 10^{3} i$

This Figure shows that the root locus (due to *k* parameter variations) is deformed by the observer dynamics. The sensitivity of the resonance frequency to stiffness variations is given by the amplitude of the imaginary axis variations. Table IV reports the poles of the observers for which simulation results are presented.

In the insert of Fig. 6, the poles of the system implemented with the observer named *inadequate* do not move compared to the ideal root locus (ideal state feedback case). If the poles of the observer are inappropriately chosen, the sensitivity of the poles to variations of k is significantly reduced. Observer poles close to the real axis, i.e., with an imaginary part close to 0, are the more appropriate to reconstruct the displacement state and to preserve the sensitivity of the system to variations of k. The socalled *fast* observer allows us to increase the resonance frequency shift ( $\omega_R = 2\pi f_R$  in Fig. 6) close to the performances theoretically expected.

### IV. EXPERIMENTAL APPLICATION FOR EXTRA-STIFFNESS CHARACTERIZATION

The previous method has been experimentally tested with the mechanical characterization of biomolecules of *fibronectin*. The physical sensor is the SNT, while the feedback controller and the observer are implemented in a dSPACE prototyping board.

#### A. Experimental Protocol

Fibronectin is a protein, 100 nm in length, of the extracellular matrix which plays a major role in cell adhesion, growth,

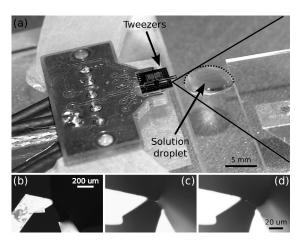


Fig. 7. Experimental setup for the trapping of *fibronectin* molecules. (a) The SNT are fixed. A droplet of *fibronectin* solution is dropped on a glass slice which is mounted on the XYZ stage. (b)–(d) Sequences of the molecule combing. Under optical microscopy, the tweezers' tips are retrieved from the solution meniscus. A bundle of *fibronectin* is released between the tips.

migration, and differentiation [33]. These molecules are used to polymerize the fibrillar matrix at the surface of the cell. We have used this property to trap by combing a bundle of *fibronectin*. After immersion of the tips in the solution, they are retrieved by the side forming a polymerized bundle of molecules (see Fig. 7).

The closed-loop systems are evaluated with *fibronectin* bundle in air, in dry condition. This stable condition was preferred to the immersed one as we can avoid any effect of liquid surface tension stability that can interfere with the sole system response. For each case, a sinusoidal reference of  $0.02\,V_{\rm AC}$  with an offset of  $20\,V_{\rm DC}$  is applied to the system and the oscillation frequencies are spanned from 1000 to  $2000~{\rm Hz}$  with a step of  $2~{\rm Hz}.$  Then, the measurement is performed with the lock-in amplifier.

Two sets of experiments have been performed with two different bundles. From the resonance frequency shifts in open loop, the stiffness of the first bundle is  $11.8~\rm N\cdot m^{-1}$ , and  $4.3~\rm N\cdot m^{-1}$  for the second one. In total, 22 resonance responses are recorded and compared.

#### B. Results and Discussions

1) Experimental Results: Fig. 8 shows the frequency responses of four different systems with and without a bundle of *fibronectin*.

In Fig. 8(a), the responses of the open-loop driven SNT are given. The dashed-line curve is the reference curve, which corresponds to the frequency response of the SNT alone driven in open loop. The solid line is the response of the tweezers with the bundle trapped in between the tips. The resonance frequency increases and the amplitude decreases due to the additional stiffness and loss contributions of the molecules. In Fig. 8(b)–(d), the results of the closed-loop systems with desired reduction ratios are of 1.1, 1.2 and 1.5.

Resonance responses are fitted with a second-order resonator model and characteristics such as the resonance frequency are extracted. The Table V sums up the resonance frequencies for the experiment with/without the first bundle for five different

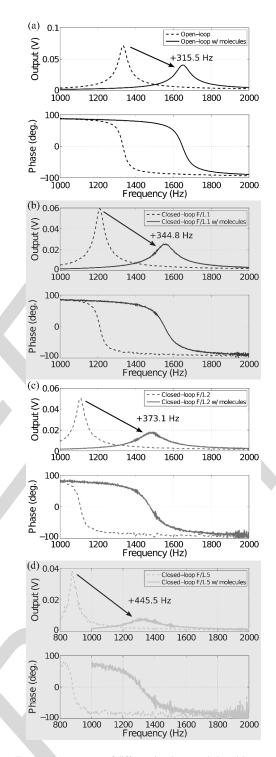


Fig. 8. Frequency responses of different implemented closed-loop systems for  $0.02~\rm V$  sinusoidal signal with an offset of  $20~\rm V$ . In dotted lines, the responses of the systems with the bundle of molecules. In plain lines, the responses of the systems without molecules. (a) Open-loop driven systems. (b) Closed-loop systems implemented with  $f_{\rm R}/1.1$ . (c) Closed-loop systems implemented with  $f_{\rm R}/1.2$ . (d) Closed-loop systems with  $f_{\rm R}/1.5$ .

systems. Through reducing the resonance frequency of the system, the measured results prove the increase of the frequency shift due to the extra-stiffness of the bundle.

Moreover Fig. 9 tends to 1) confirm with second set of experiments, the enhancement of the sensitivity through the proposed

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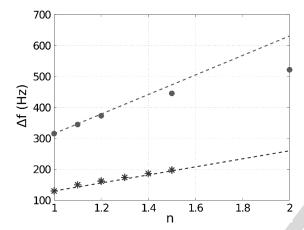
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TABLE V EXPERIMENTAL RESULTS FOR THE BUNDLE 1 (RELATED TO FIG. 8)

System (n)	Freq. w/o bundle (Hz)	Freq. w/ bundle (Hz)	Freq. shift (Hz)
Open loop (1.0)	1333.6	1649.1	+315.5
Closed-loop (1.1)	1211.1	1555.9	+344.8
Closed-loop (1.2)	1109.1	1482.2	+373.1
Closed-loop (1.5)	881.9	1327.4	+445.5
Closed-loop (2.0)	653.9	1175.6	+521.7



Synthesis of the experimental results. In abscissa, is informed the reduction factor (n) applied to the resonance frequency for the closed-loop system. The shift  $\Delta f$  is the resonance frequency difference due to the presence of the bundle of molecules in between the tips. The dotted lines represent the theoretical improvement expected. The two sets of results obtained with two different bundles are plotted. The green dots correspond to the experiment of Fig 8 and TableV (bundle of  $11.8 \text{ N} \cdot \text{m}^{-1}$  stiffness). The blue stars corresponds to the second bundle of  $4.3~\mathrm{N}\cdot\mathrm{m}^{-1}$  stiffness.

feedback strategy and 2) show a good agreement with the theory. Small differences are discussed in the following section.

2) Discussion: The synthesis of Fig. 9 demonstrates the enhancement of the frequency shifts in agreement with the theory. However, the enhancement for closed-loop system with a reduction factor of 2.0 is lower than expected, and the implementation for a higher reduction factor has not been achieved. In this case, the system tends to become unstable. We mainly associated this phenomenon to the observer implementation and to the presence of some delays in the feedback loop.

We have demonstrated previously that observer's dynamic changes the root locus of system's poles, the behavior of the closed-loop system under parameter variations and impacts the performances of the method. Furthermore, delays have been characterized due to D/A and A/D converters dead times and to the phase shift brought by the current amplifiers. Current works are devoted to the precise characterization of such delays and to their integration in the control design process.

#### V. CONCLUSION AND FUTURE WORK

The present study investigates the relevance of a feedback approach in the improvement of the performances of MEMS tweezers for the detection and the mechanical characterization of biological molecules. An improvement of 50% on the sensitivity

to mechanical stiffness variations has been reached theoretically and experimentally on *fibronectin* bundles through the emulation of a more compliant system. The stiffness have been brought 410 down to 10.5 N/m instead of originally 24.5 N/m.

The main drawback of this approach is the sensitivity of 412 the observer to the stiffness variations. Unusually, the poles of the observer are uncontrollable resulting from the separation 414 principle. During the sensing of the molecules stiffness, the 415 separation principle is not satisfied anymore and the observer 416 poles can become unstable. We are actually working on selfscheduled observers aiming at adapting the observer dynamics 418 with respect to sensed parametric variations. It should solve the main drawback of this promising approach and allows us to go one step further into the theoretical closed-loop performances.

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